

# *Origins and Dynamics of Wide Gage Derailments*

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# Why A Presentation on Wide Gage Derailments?

- Most used cause code in the FRA Accident Database
- ~20% of all track caused derailments attributed to T110, T111 Wide Gage
- Many derailments listed as “wide gage” incorrectly blamed on “wide gage”.
- Other Causes such as mechanical and train handling often not listed.



*“Wide Gage description possibly  
the least understood of all  
derailment causes”*

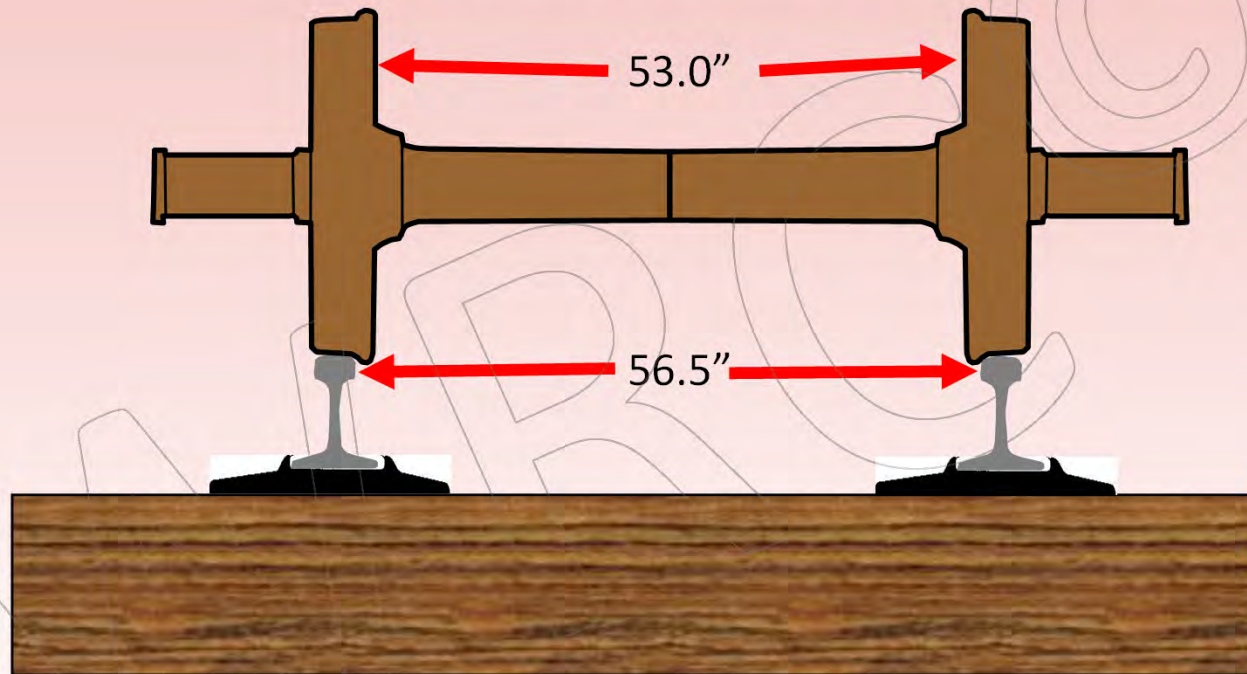


*“The true root causes of wide gage are generally not reported. Not all ‘wide gage’ derailments are track caused. Many ‘wide gage’ derailments have mechanical and train operations root causes”*

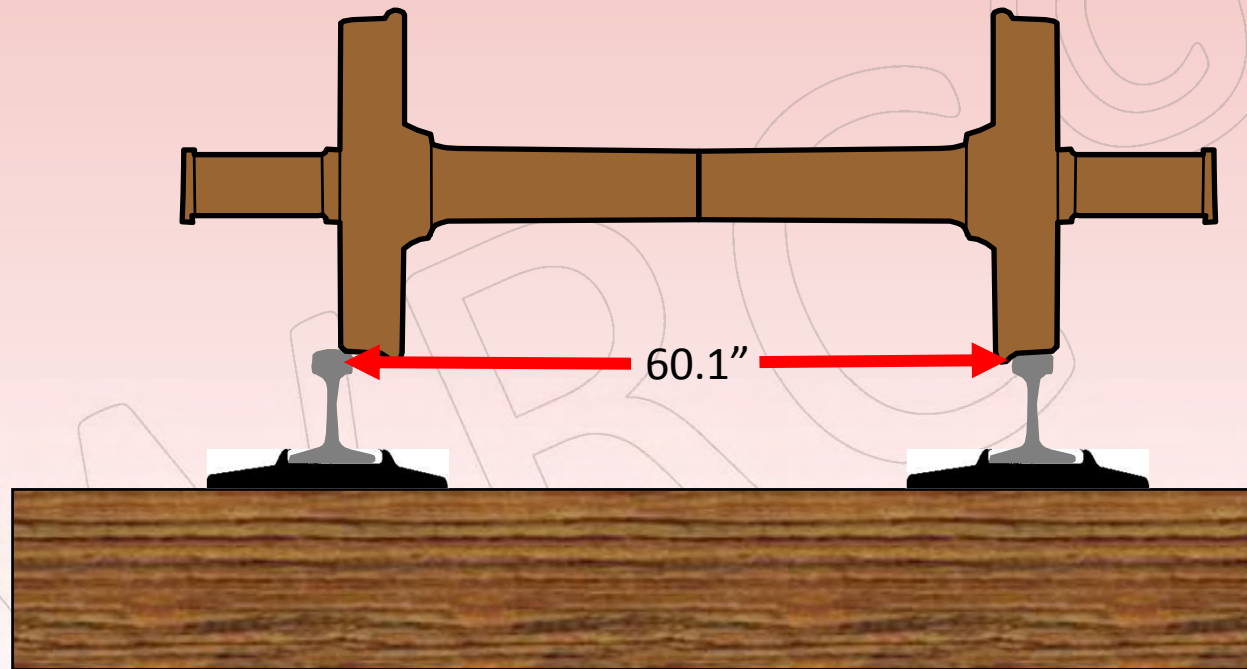




Nominal gage of 56.5" with  
53" Wheel back to back



Point of incipient wheel drop  
Due to widening of gage

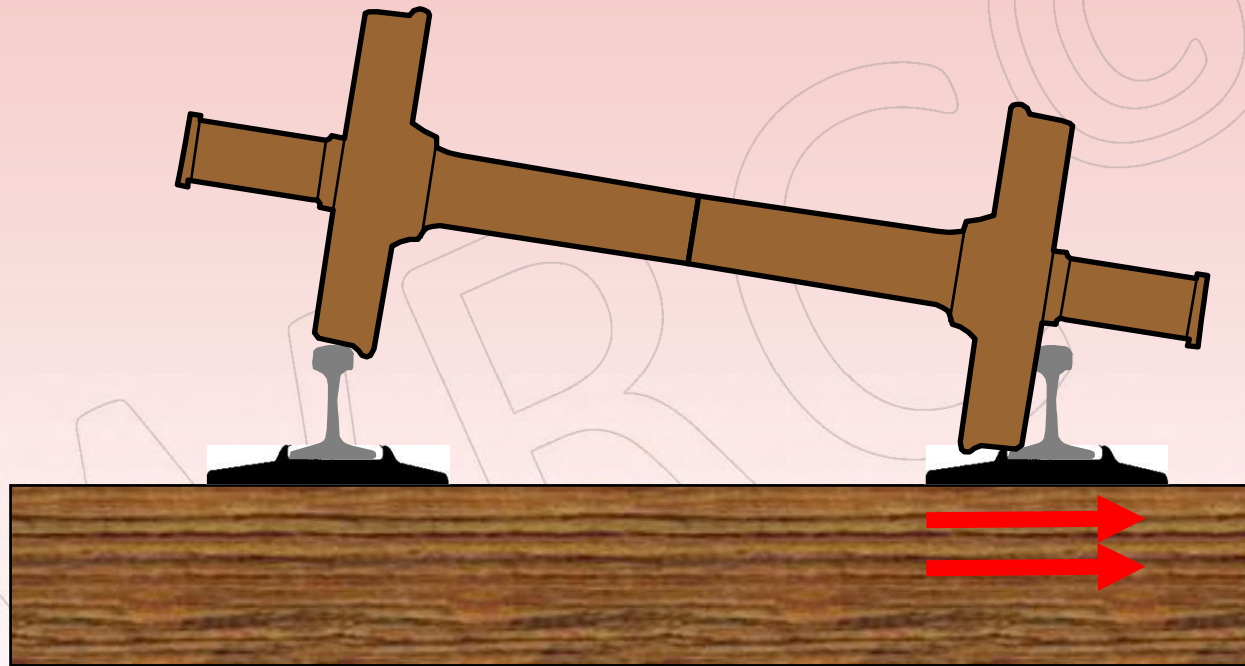


Wheel/Rail Geometry with 60.1" Gauge

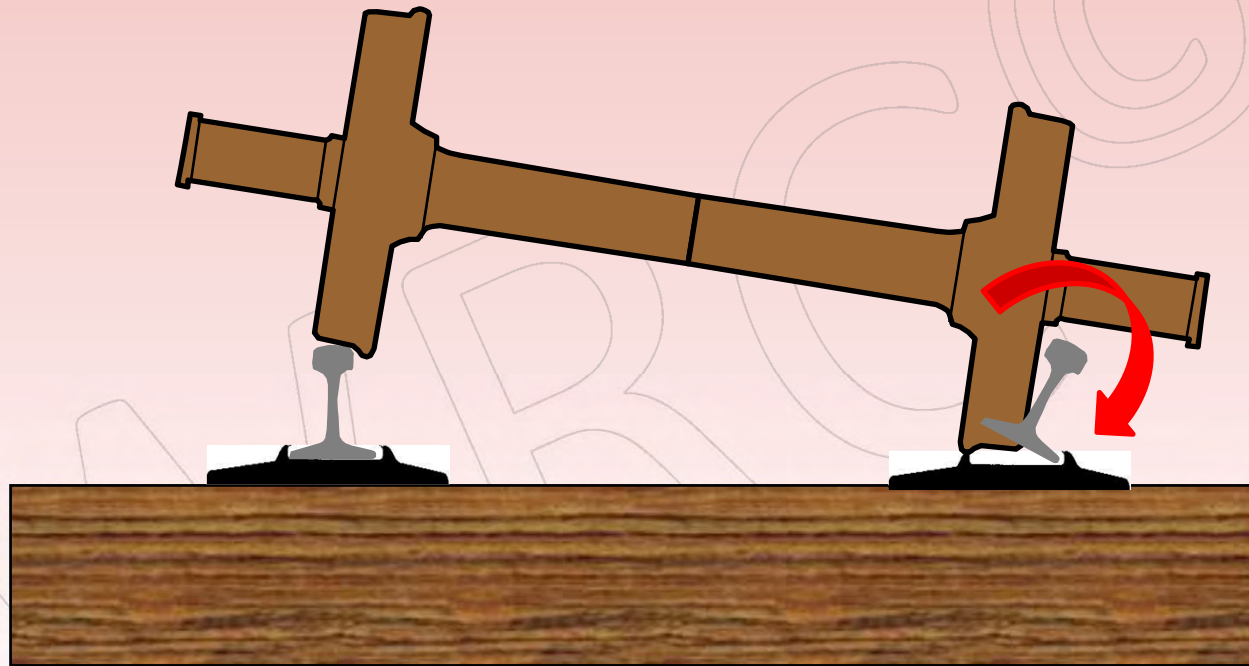
Note: This is gauge point at which wheel drop becomes likely due to radius of wheel intersecting radius of rail head



Wide Gage due to lateral  
Translation of rail



Wide Gage due to rail  
Rolling outward





# Wide track gage is caused by:

- Lateral translation of one or both rails, due to:
  - Deformation of wood fibers holding spikes; elongated spike holes; spike kill
  - Broken screw spikes or cut spike fasteners
  - Worn shoulders on tie plates
  - Gage face wear on the rail
- Rail rolling outwards, due to:
  - Differential tie plate cutting; rail cant
  - Loose fasteners; weak spiking; insufficient spiking
  - Poor wheel contact geometry toward field side of rail head
  - Hollow worn wheels contacting field side of rail head
- A combination of lateral translation and rail roll



## How does Gage Widen?

- Progressive failure (fatigue) over weeks or months
  - *“Death by a thousand small cuts...”*
    - ✓ Spike kill/ tie deformation over time
    - ✓ Fastener failure
    - ✓ Differential plate cutting; rail cant
    - ✓ Geometry defect
- *Singular catastrophic event*
  - *“Death by a single blow to the head...”*
    - ✓ Bypassed couplers during switching
    - ✓ Excessive buff (compressive) force
    - ✓ Use of non-aligning couplers
    - ✓ Truck warp; locked bolster on truck



# What causes gage to widen?

- Excessive lateral pressure against the rail due to:
  - Incorrect curve elevation
    - Insufficient elevation causing pressure of high rail
    - Excessive elevation causing pressure on low rail
  - Pre-existing wide gage allowing greater wheelset angle of attack
  - Horizontal alignment kinks causing increase in flanging force
  - Lack of rail lubrication, particularly top of rail extremely dry
  - Poor steering due to poor wheel – rail contact geometry



# What causes gage to widen (con't)?

- Excessive lateral pressure against the rail due to:
  - Excessive compressive buff force in train
    - Excessive dynamic braking
    - Excessive independent braking
    - Excessive shoving amperage
    - Bypassed couplers
  - Stiff truck condition; excessive turning moment locking up bolster rotation
    - Centerbowl not lubricated
    - Centerbowl rim contact or fouling
    - Tight side bearings
    - Improper centerplate repairs
  - Warped truck condition
    - Lack of truck squaring force due to wedge rise; bolster pocket wear; column plate wear





Let's review how our maintenance strategies and operating procedures can contribute to increased lateral pressure against both the high and low rail of curves...



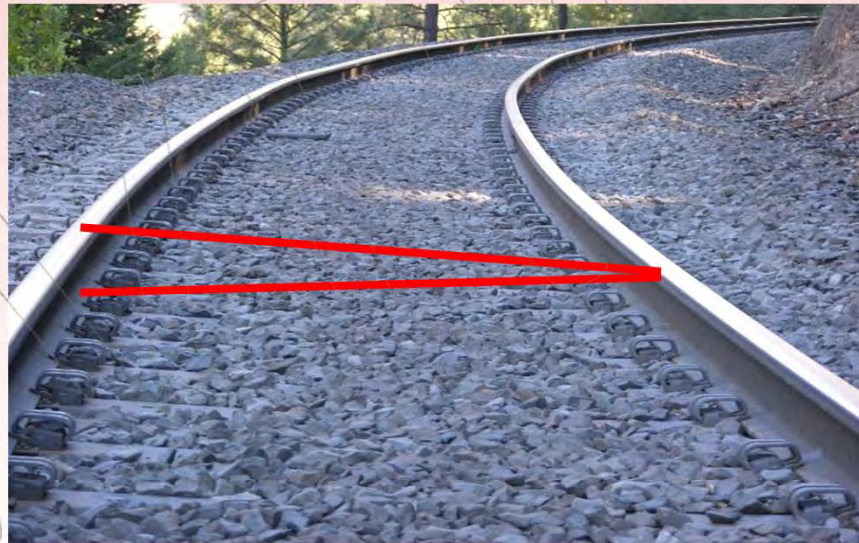
# Curve Superelevation Issues

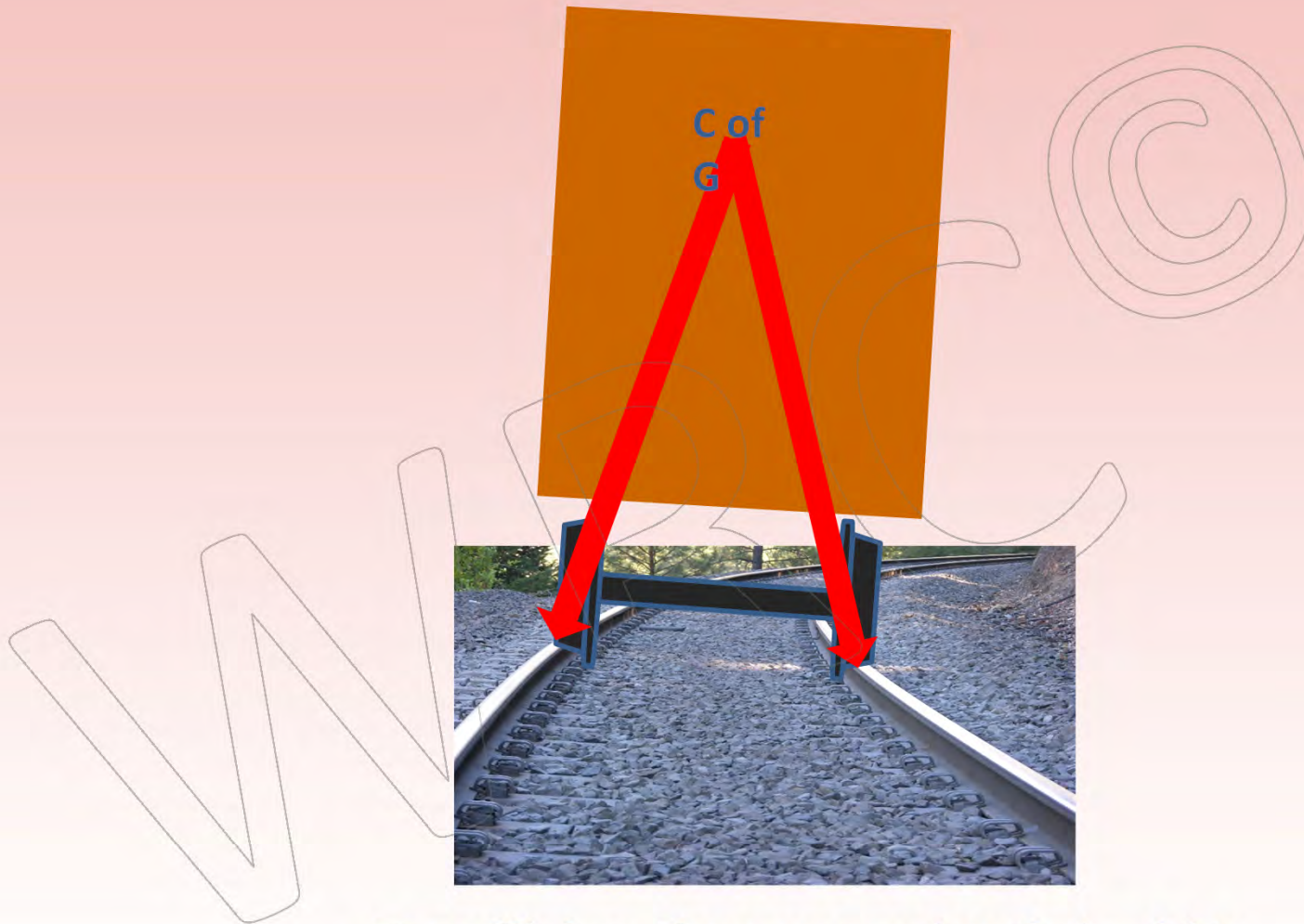




## Superelevation in a curve

Normally, between 0.0 to 6.0 inches of elevation is added to outer rail to counterbalance effects of centrifugal forces based on normal train speeds.





At equilibrium, There are equal vertical weights on both high and low rail.





# To determine Equilibrium Speed for Curve

$$\text{Eq. Elevation} = .00067 (D) (V^2)$$

For 5 degree curve; 30 MPH

$$\text{Eq. El.} = .00067 (5) (30^2)$$

$$\text{Eq. El.} = .00067 (5) (900)$$

$$\text{Eq. El.} = 3.00 \text{ "}$$

$$1 \text{ " Unbalance} = 3.00 - 1.00 = 2.00 \text{ "}$$



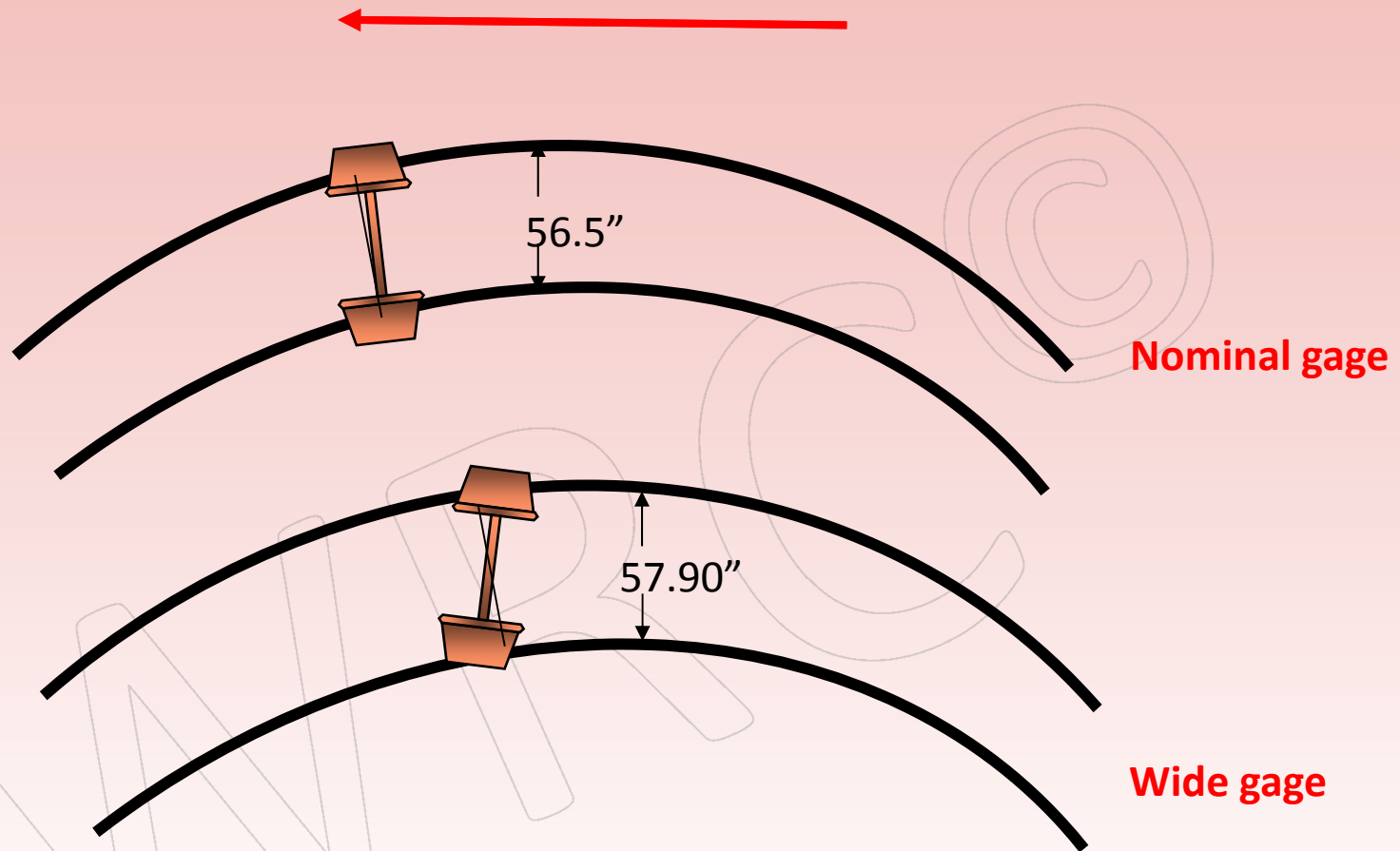
# Superelevation and adverse lateral forces

- **Superelevation is added to curves to allow trains to operate at normal track speed without placing abnormally high lateral and vertical forces against the high rail.**
- **Insufficient elevation can lead to excessive lateral pressure against the high rail in curves**
- **Excessive elevation places adverse vertical forces on the low rail of curves. Excessive vertical forces on the low rail of curves also increases the development of large lateral creep forces toward the field side.**



# Pre-Existing Wide gauge



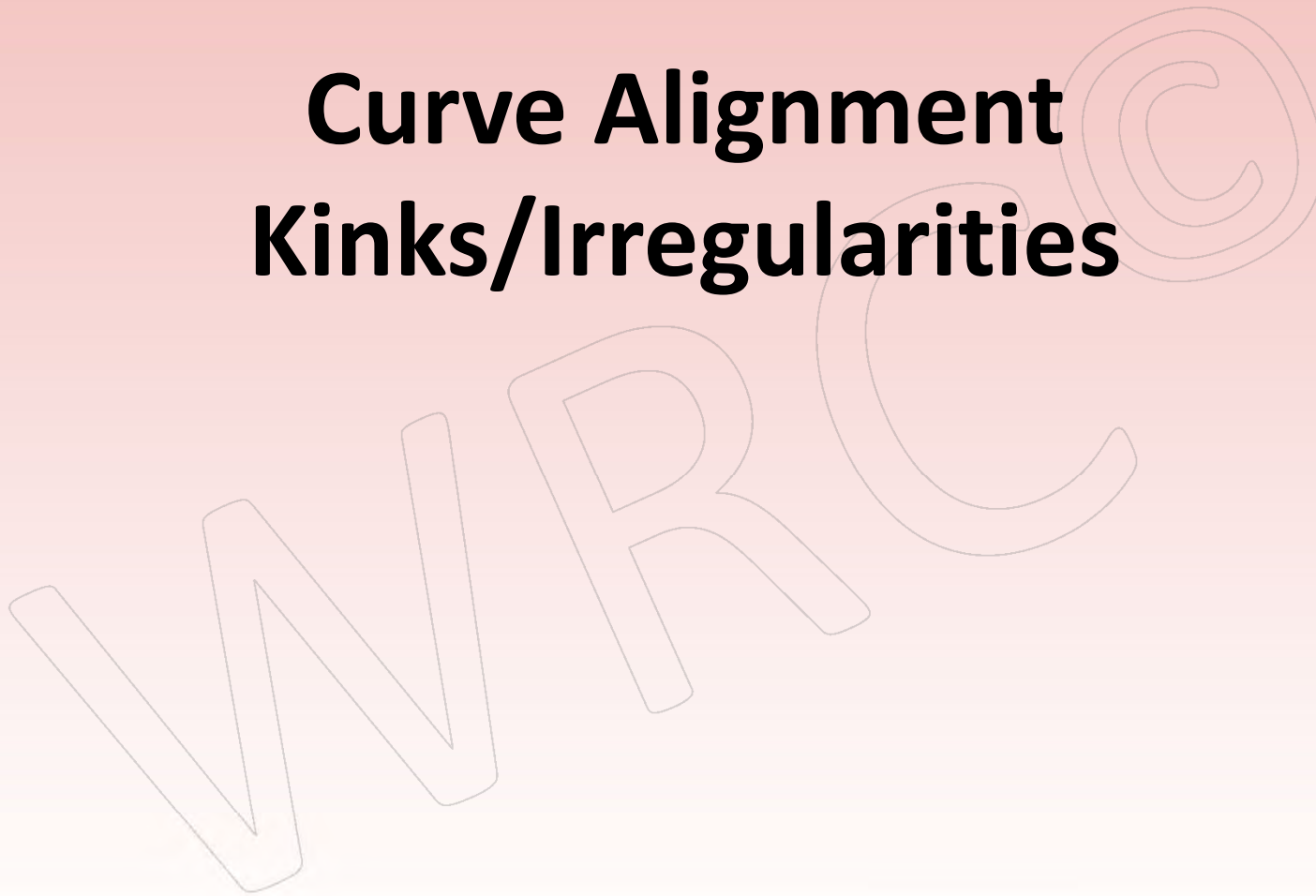


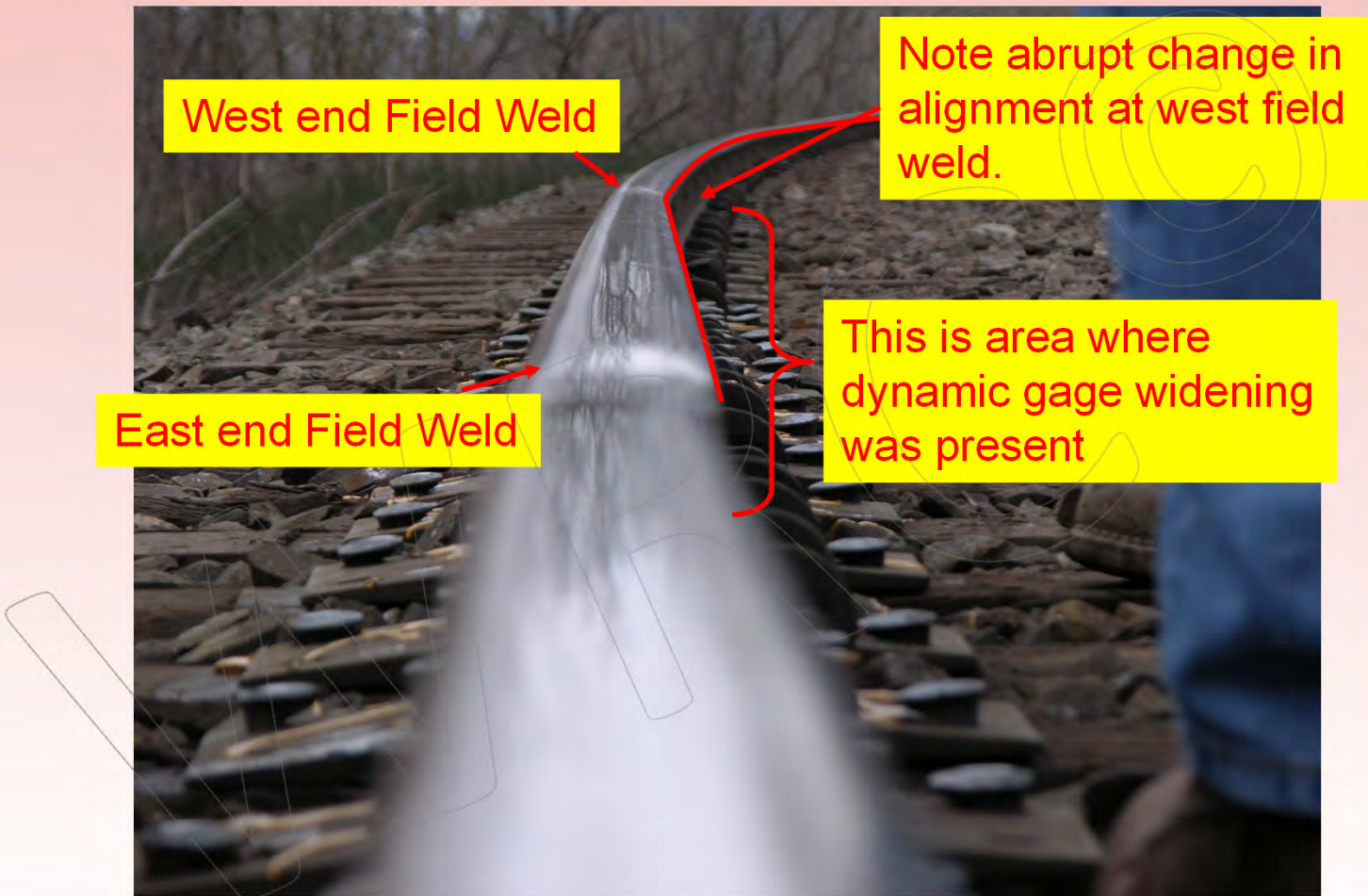
**Pre-existing Wide Gage can allow greater wheelset angles of attack;  
Greater angles of attack can cause greater lateral creep forces.  
Wide gage also can lead to poor wheelset steering and adverse contact geometry**

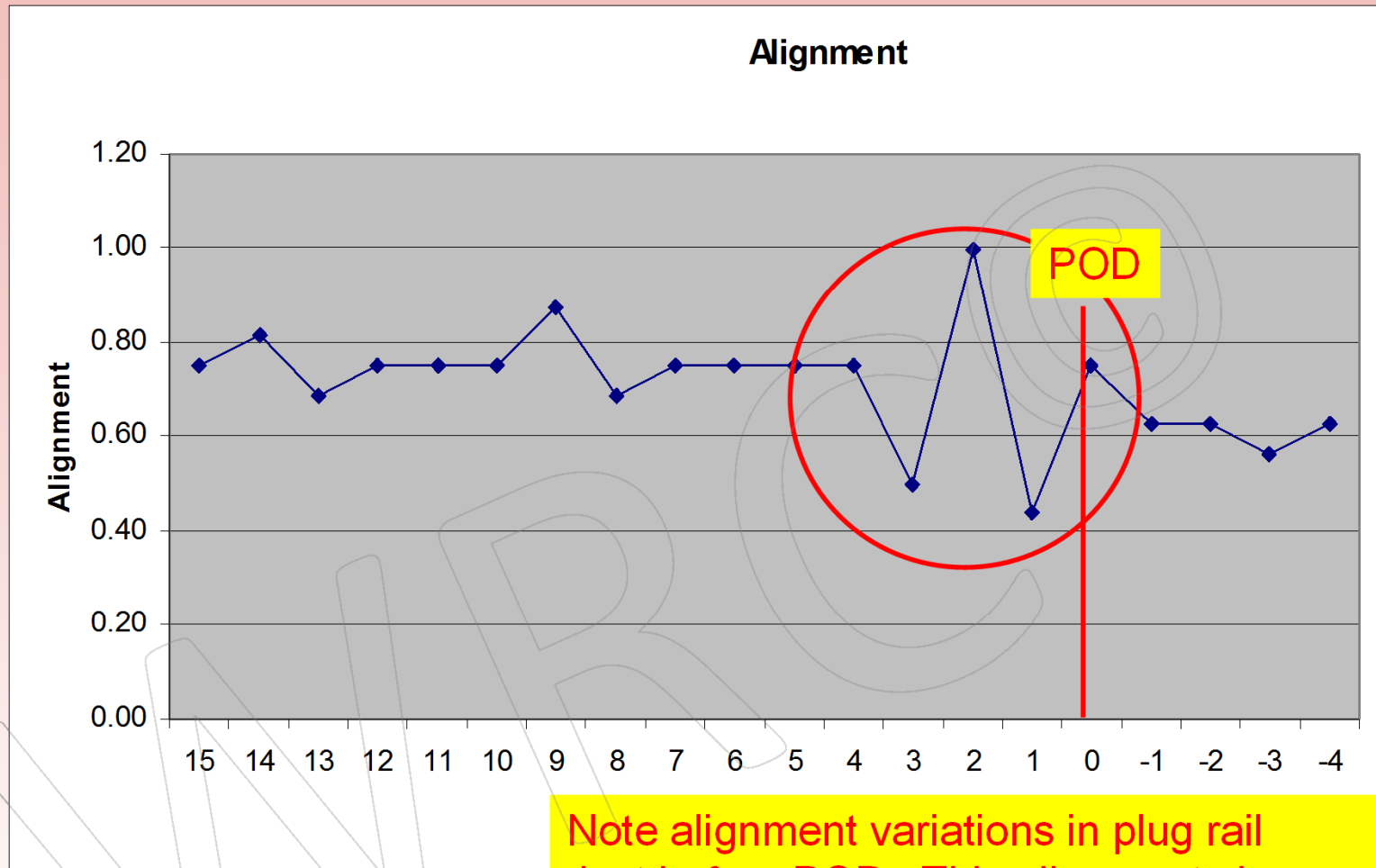




# Curve Alignment Kinks/Irregularities



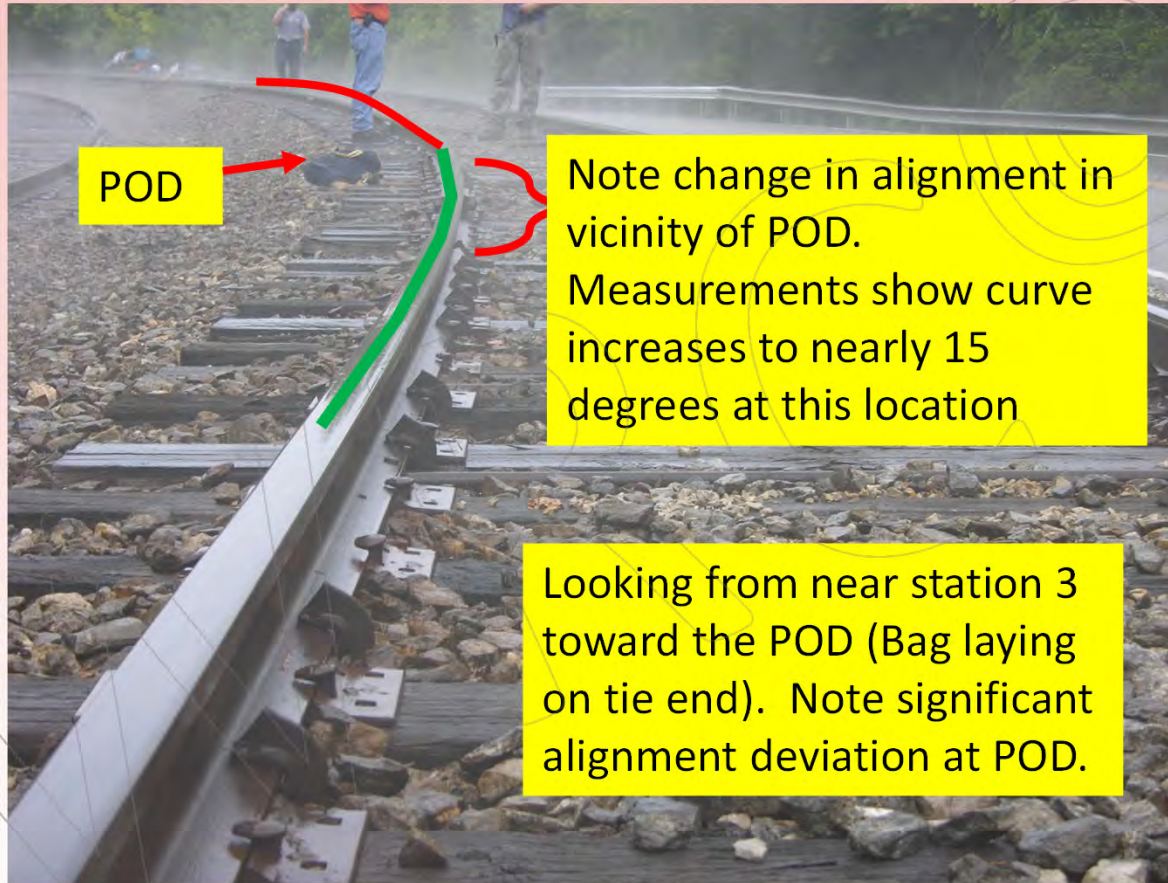




Note alignment variations in plug rail just before POD. This alignment change is evident in photos in report.

This chart prepared by using the actual ordinates from a 31 ft. chord





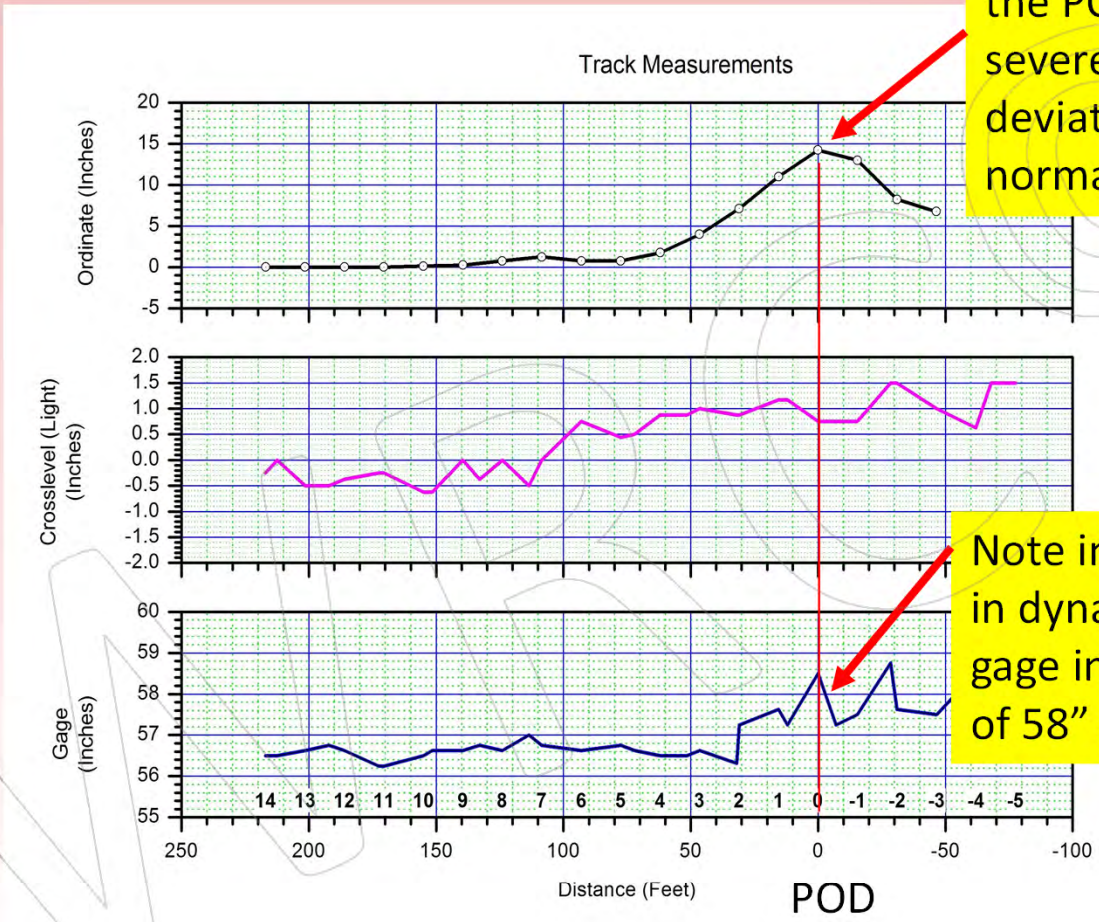
POD

Note change in alignment in vicinity of POD. Measurements show curve increases to nearly 15 degrees at this location

Looking from near station 3 toward the POD (Bag laying on tie end). Note significant alignment deviation at POD.







Note sharp increase in degree of curve at the POD due to severe alignment deviation from normal.

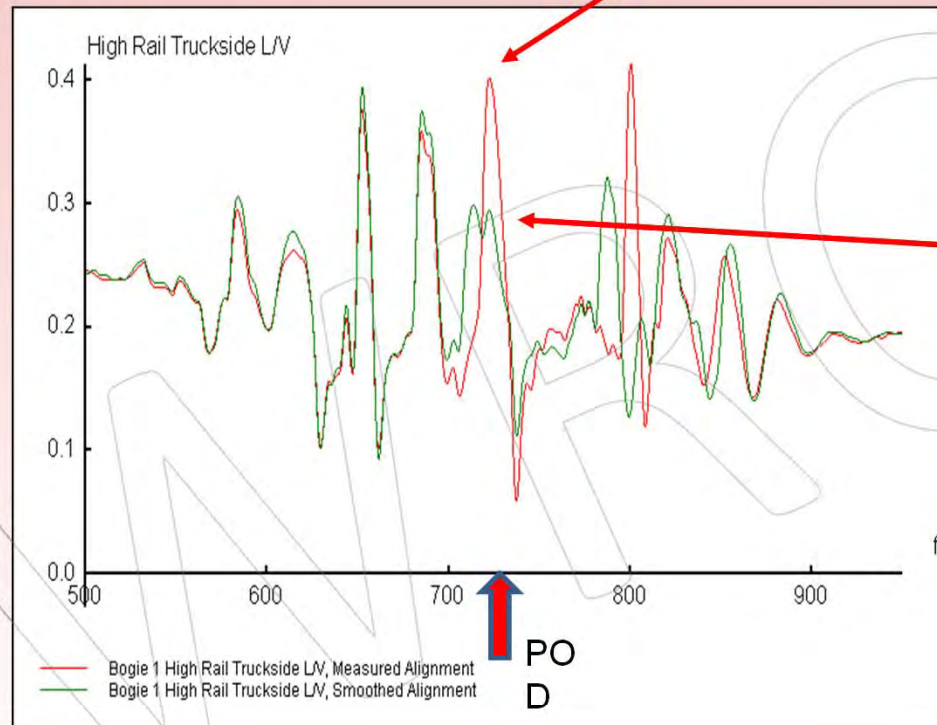
Note increase in dynamic gage in excess of 58" at POD





## Truckside L/V Results: Measured vs. Smoothed Alignment

Note, peak L/V Ratios at the POD



Truck Side L/V at POD:

Measured  
Alignment = 0.40

Smoothed  
Alignment = 0.29



# Inadequate top of rail lubrication

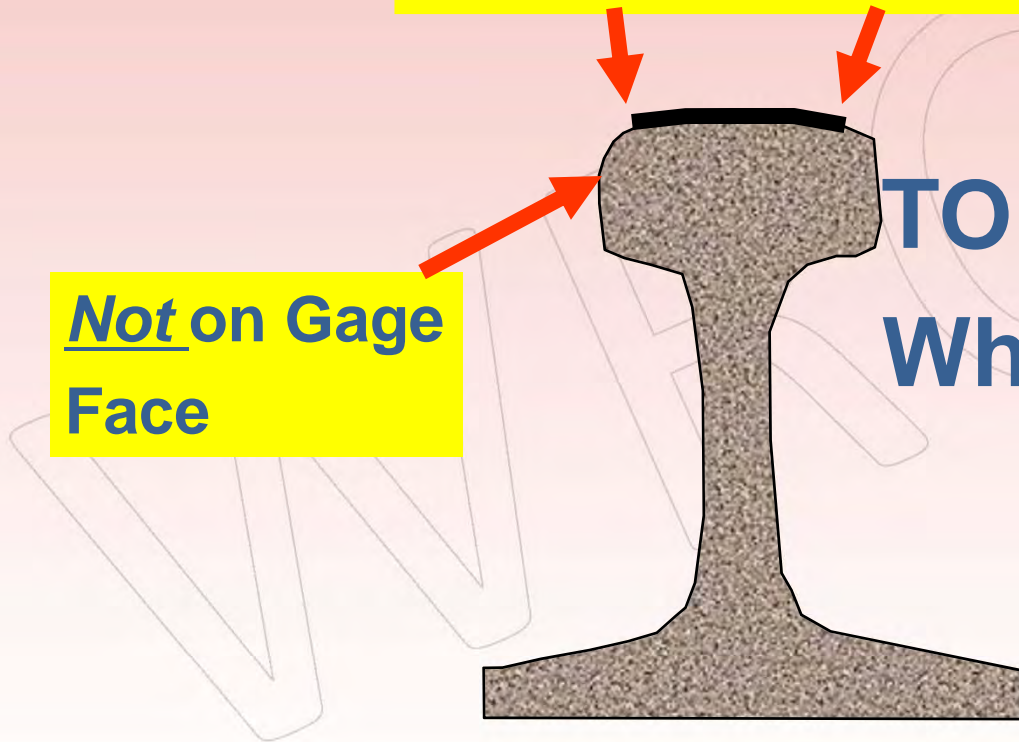
WVRC ©



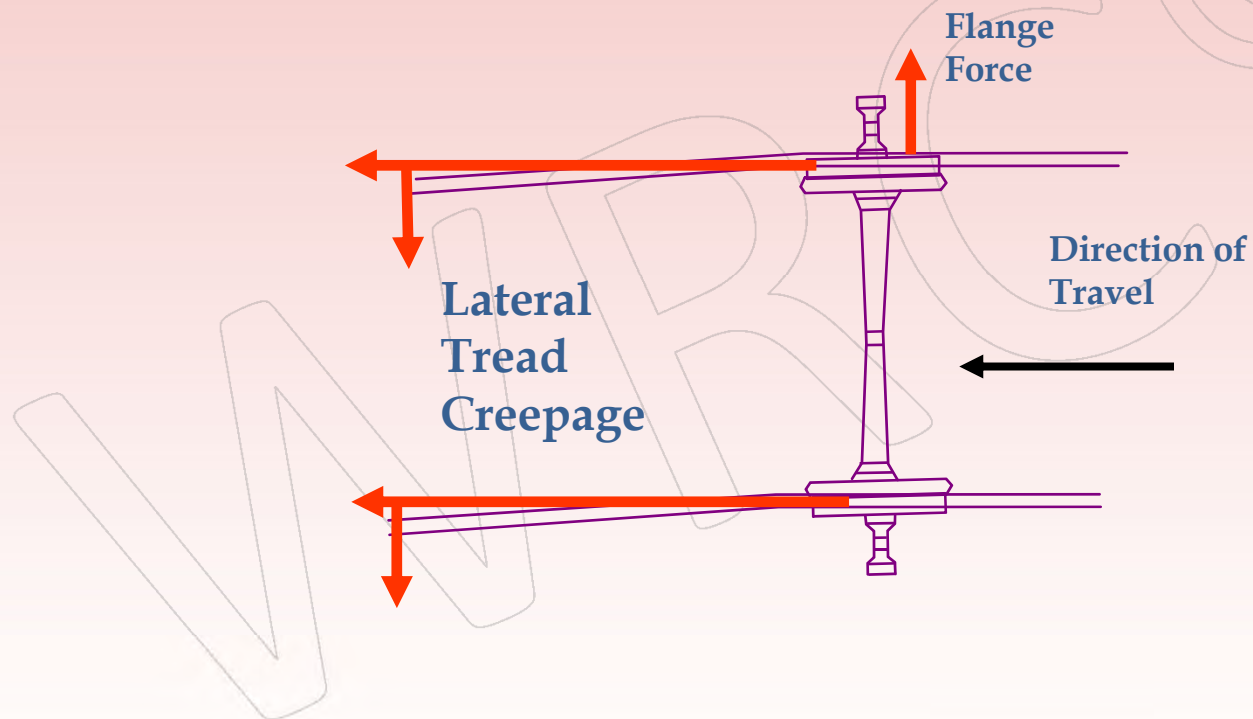
Top of rail friction control occurs in this area of rail head

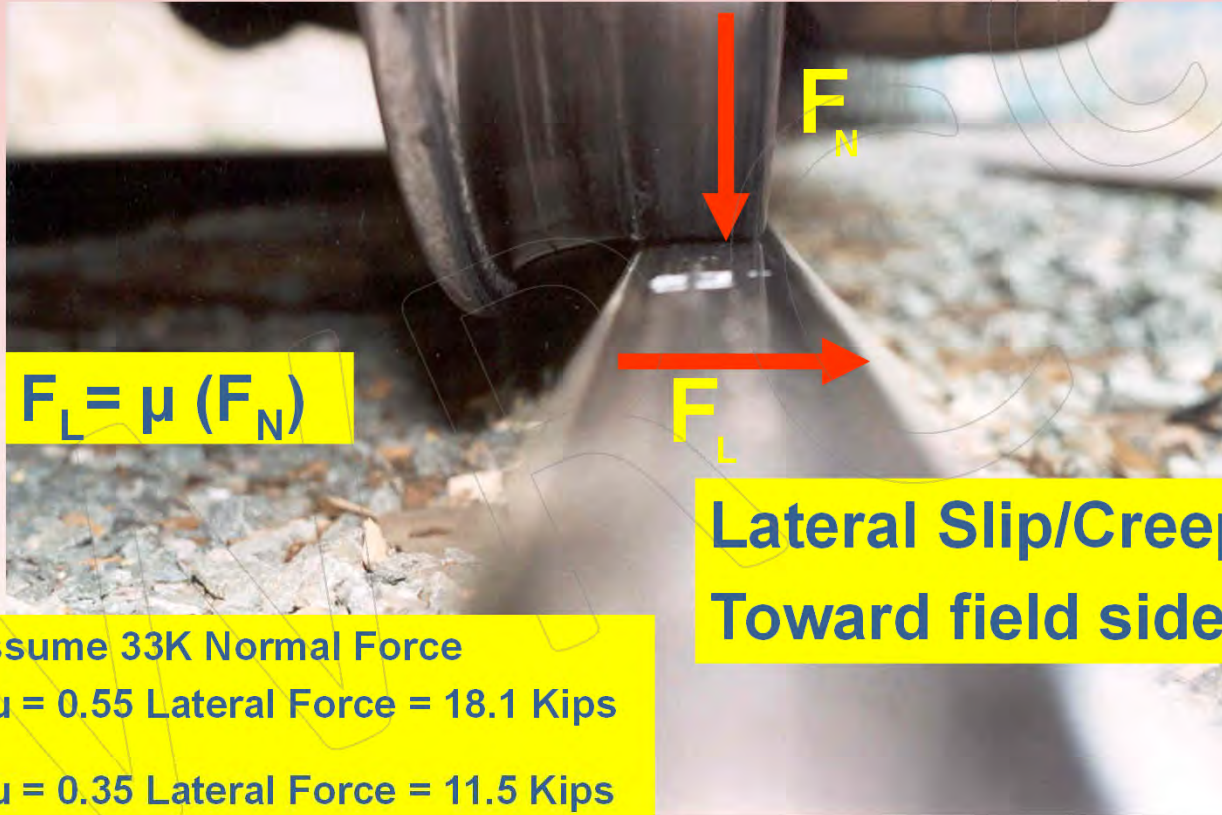
Not on Gage Face

TOR  
What is it???



## How do we develop lateral forces on top of rail toward the inside of a curve?



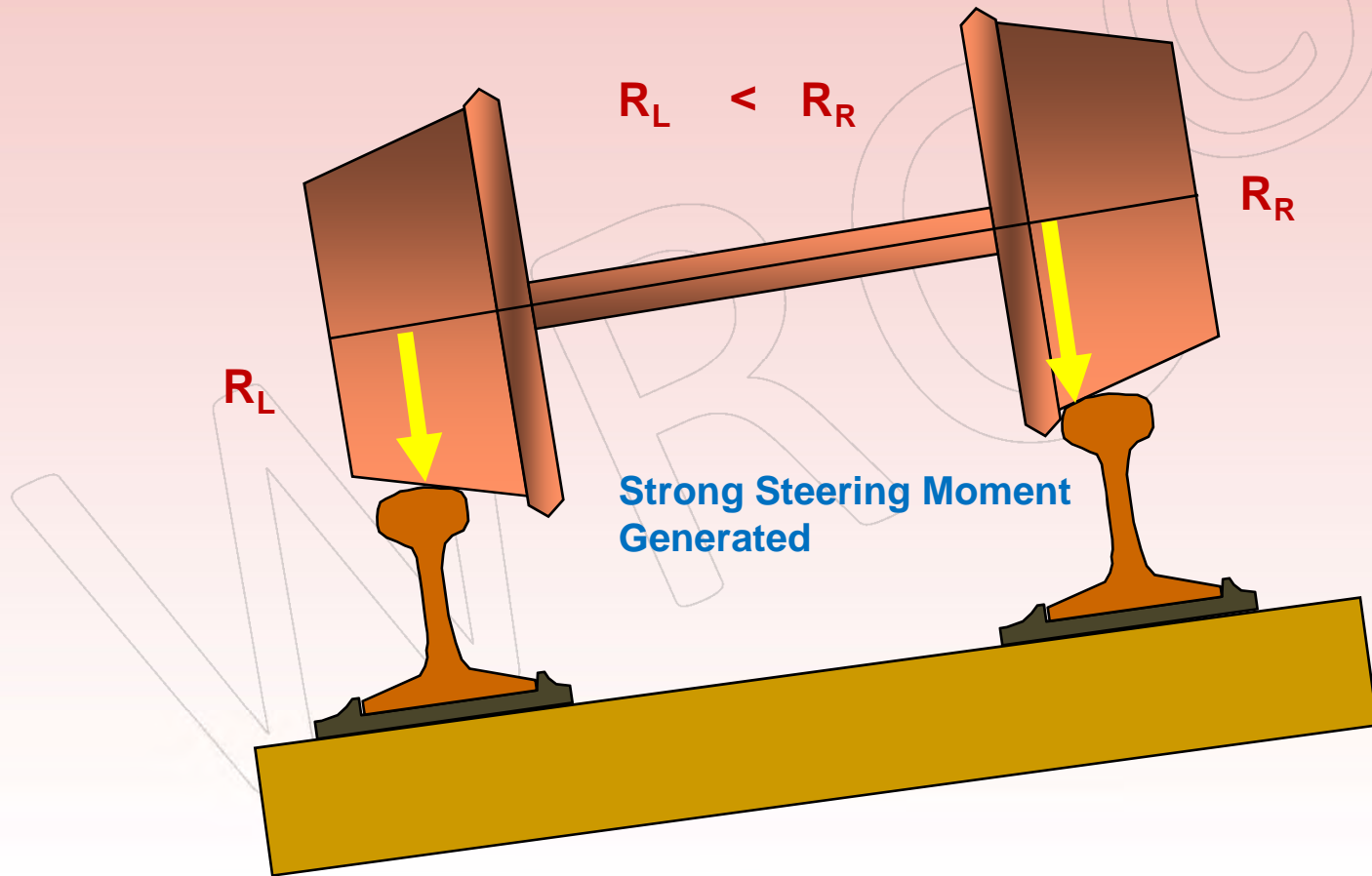




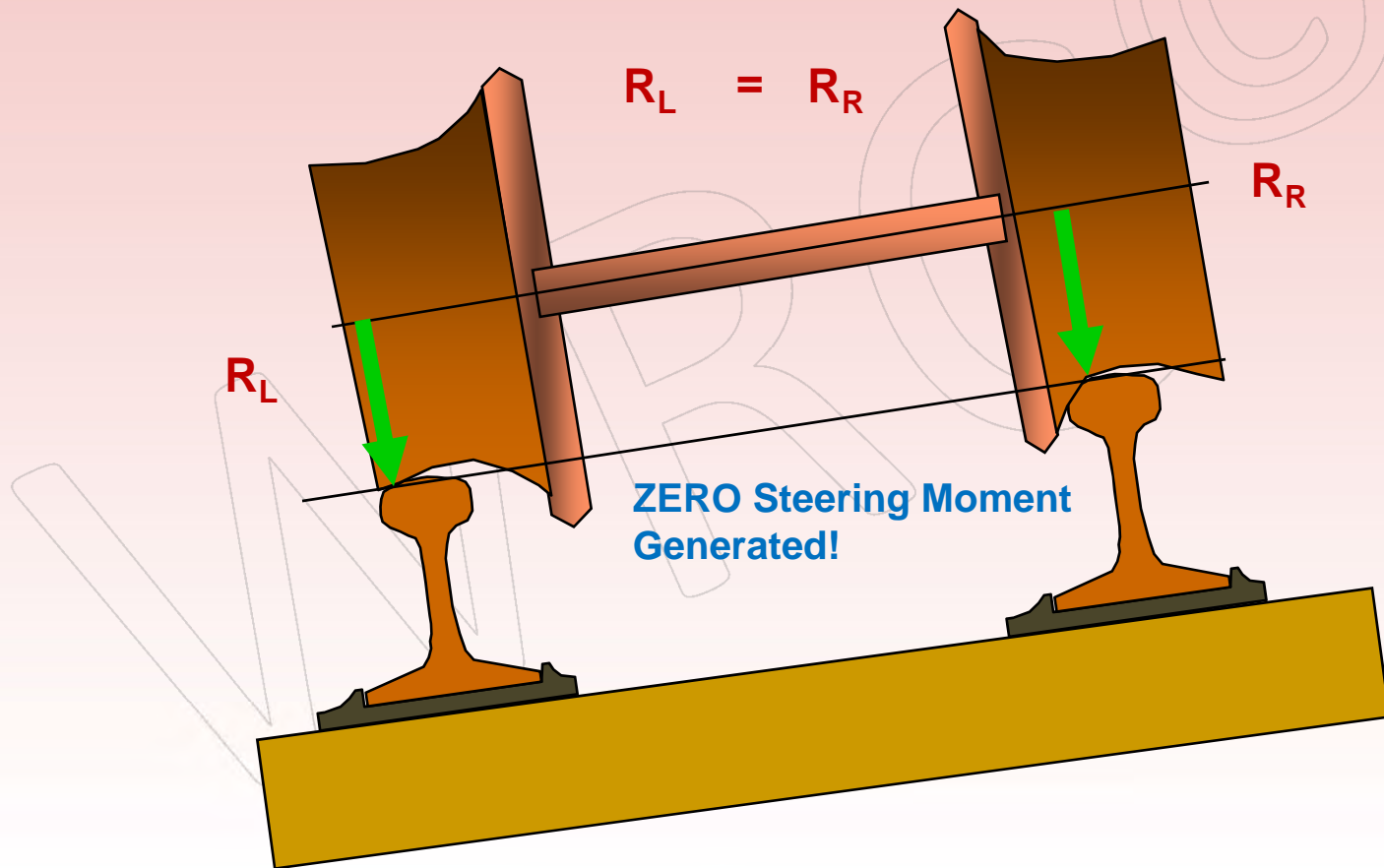
# Poor wheel rail contact geometry



## Normal Curving



## Hollow Wheel Curving

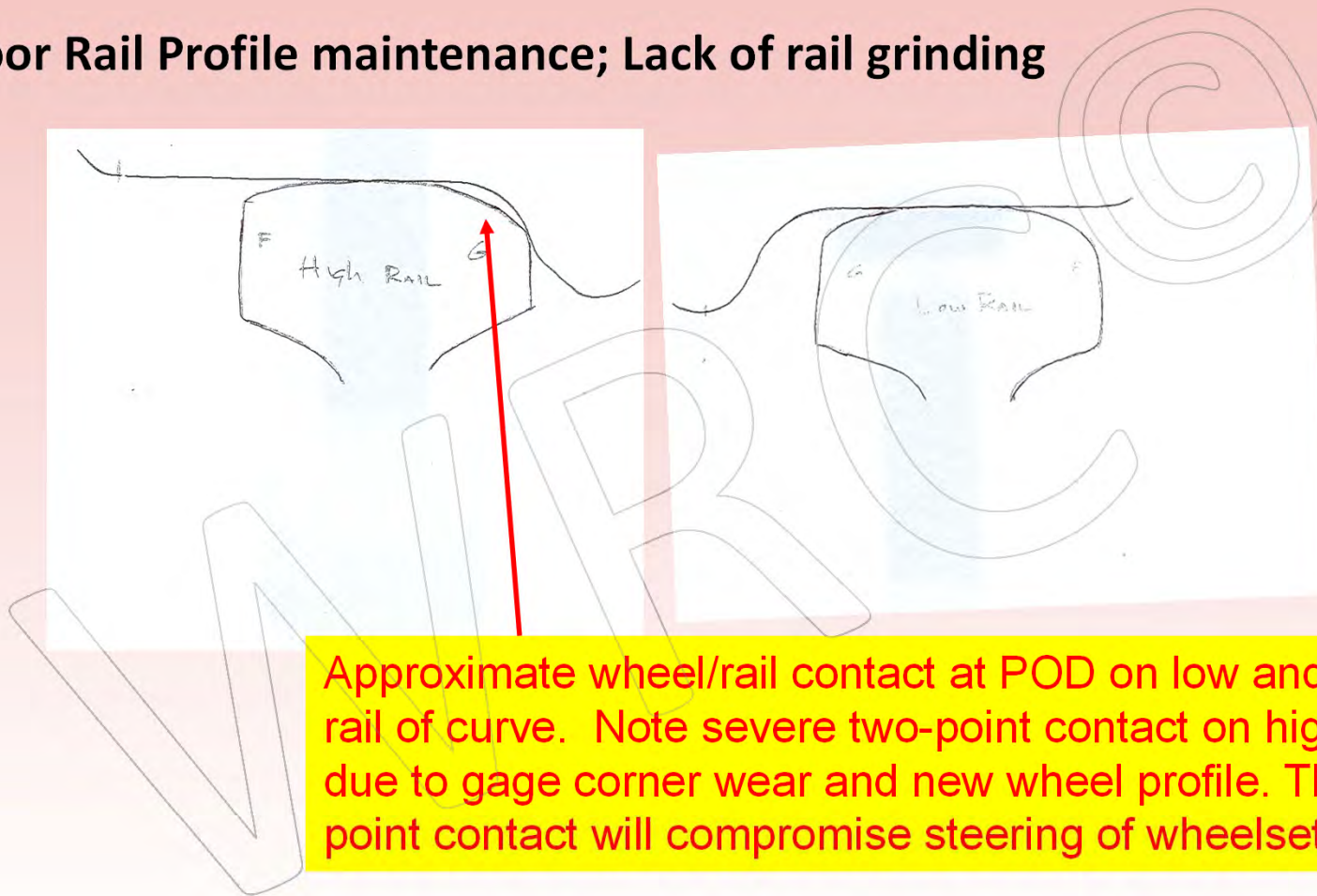




**Hollow Worn Wheels**



## Poor Rail Profile maintenance; Lack of rail grinding

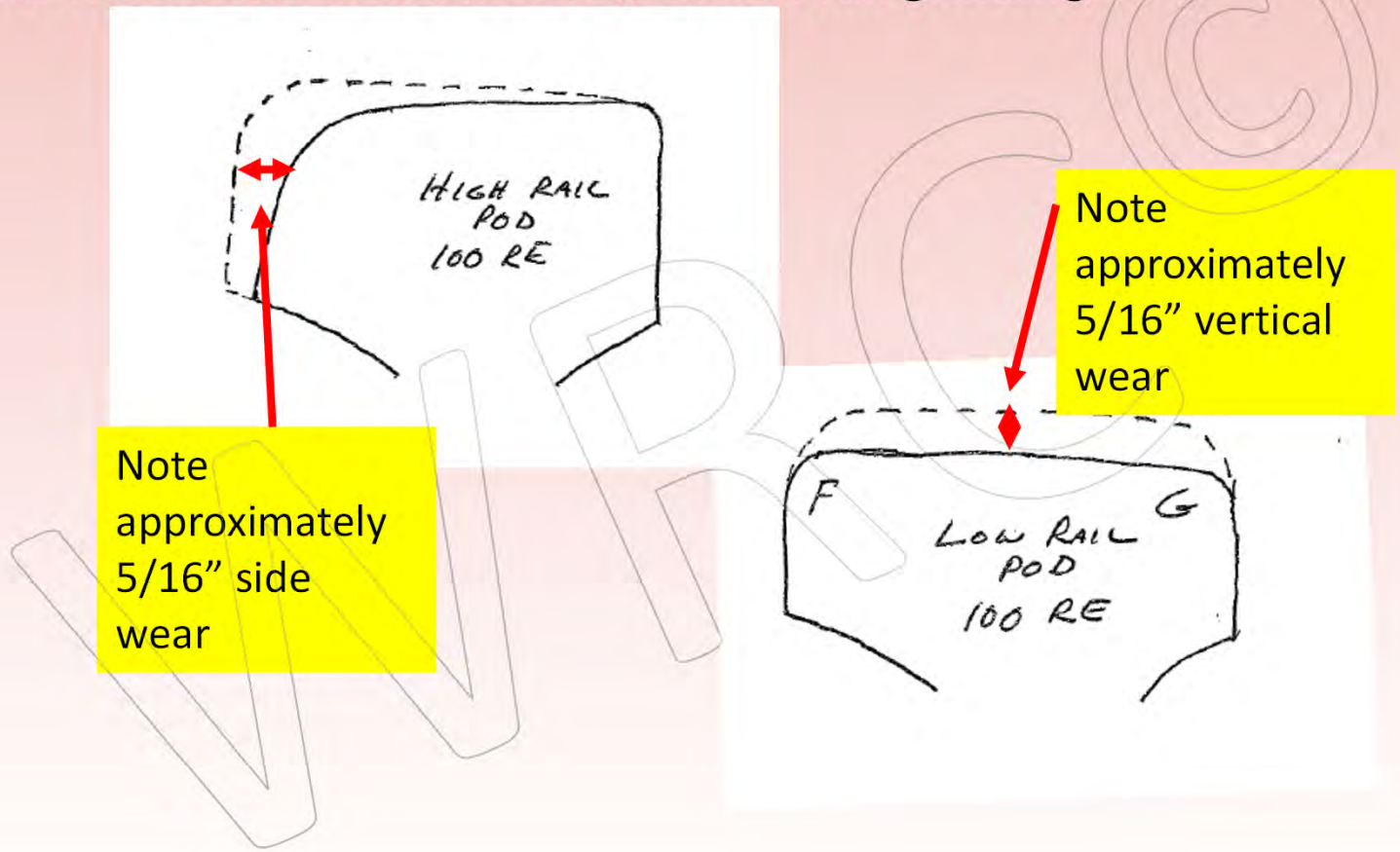


Approximate wheel/rail contact at POD on low and high rail of curve. Note severe two-point contact on high rail due to gage corner wear and new wheel profile. This two point contact will compromise steering of wheelset.





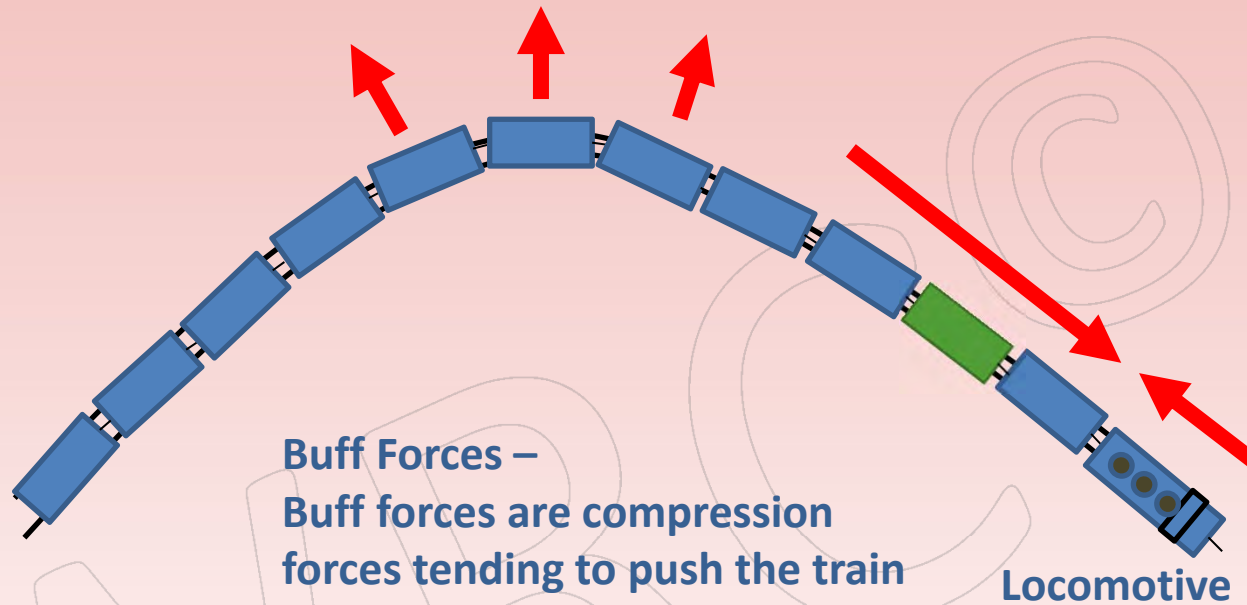
## Poor Rail Profile maintenance; Lack of rail grinding



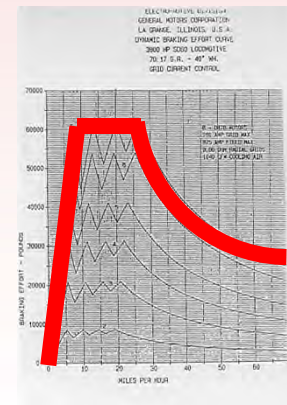
# Excessive compressive buff force in train

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**Buff Forces –**  
 Buff forces are compression forces tending to push the train outward in a curve. Often called “jackknifing” forces



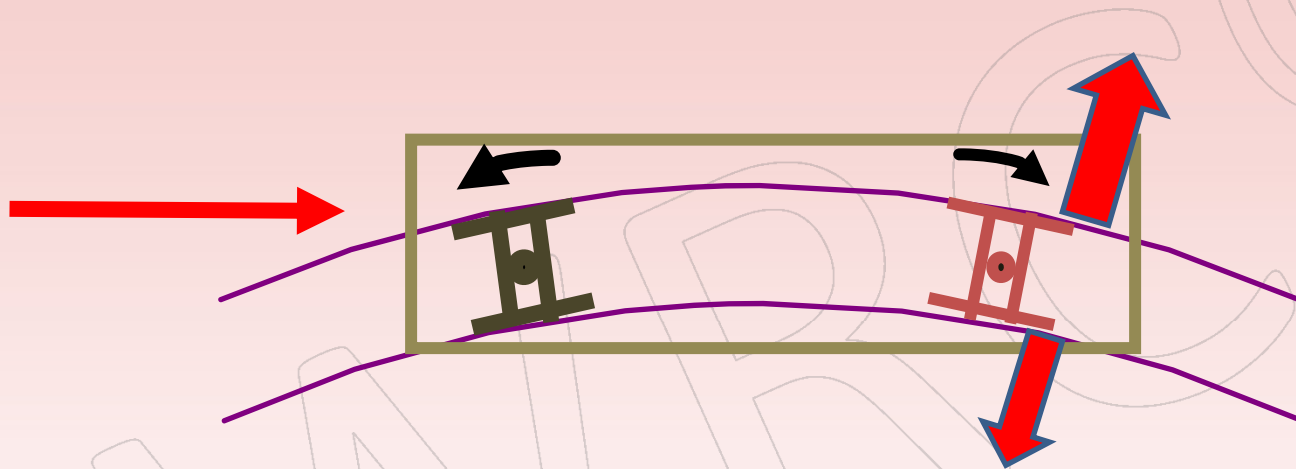
# Stiff truck condition; bolster not steering

WVRC





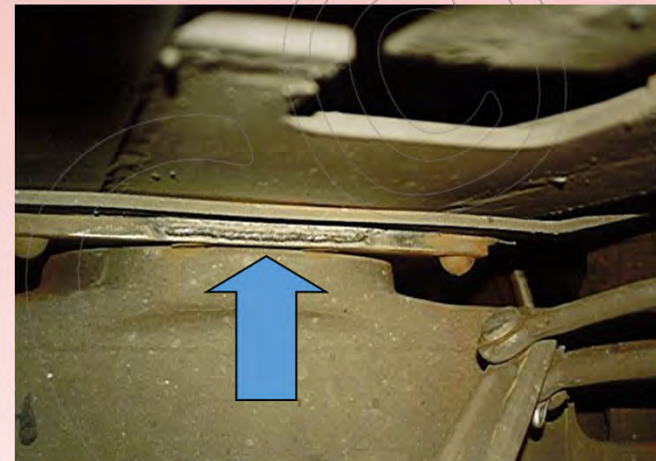
## Truck Bolster Position During Curve Negotiation



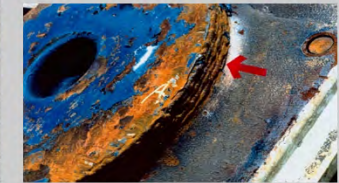
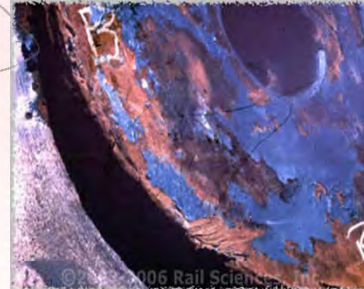
**Lead and Trailing Truck Bolsters  
must pivot in opposite  
directions**



- ✓ Improper centerplate repair
- ✓ Lack of bowl lubrication
- ✓ Insufficient side bearing clearance
- ✓ Fouling between bowl and rim



**Centerplate Weld Repair**



5" - 5 1/16"  
Set Up Ht.



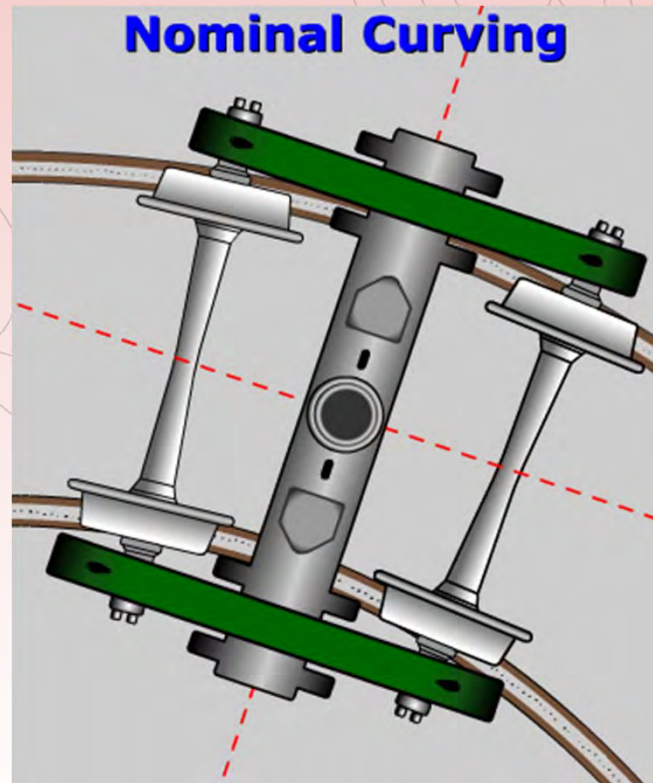
# Truck Warp Condition

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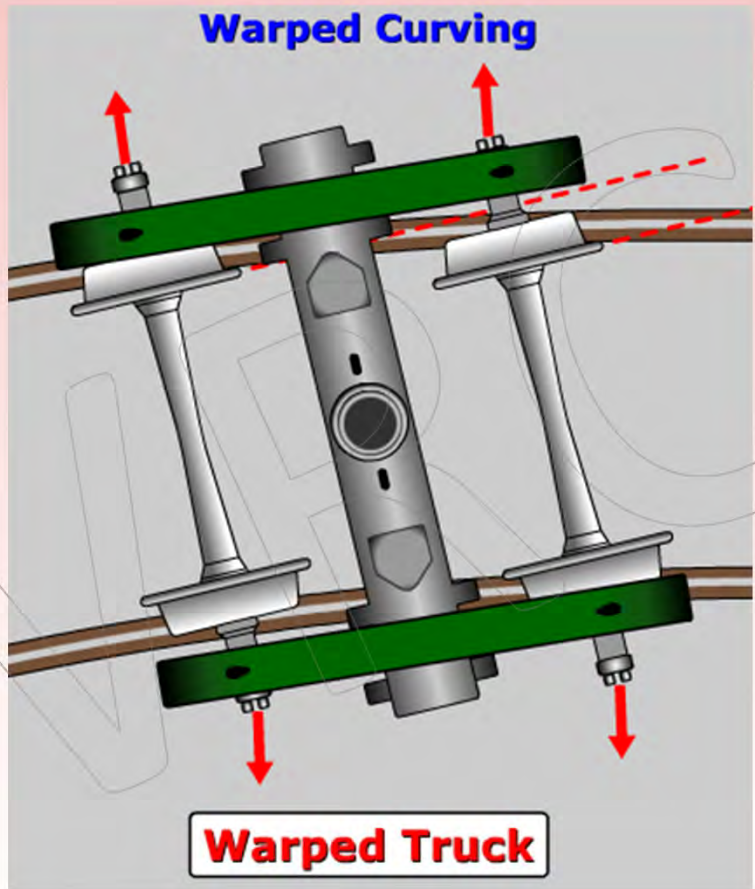


# Truck Warp Restraint

Ideally, a truck should remain “Square” during curving to allow radial alignment of wheelsets with curve







OK, we now understand that progressive and catastrophic gage widening is caused by excessive lateral force against both the high and low rail of curves.

Now let's review various track conditions that precipitate widening of the gage....



# Wide track gage is caused by:

- Lateral translation of one or both rails, due to:
  - Deformation of wood fibers holding spikes; elongated spike holes; spike kill
  - Broken screw spikes or cut spike fasteners
  - Worn shoulders on tie plates
  - Gage face wear on the rail
- Rail rolling outwards, due to:
  - Differential tie plate cutting; rail cant
  - Loose fasteners; weak spiking; insufficient spiking
  - Poor wheel contact geometry toward field side of rail head
  - Hollow worn wheels contacting field side of rail head
- A combination of lateral translation and rail roll



# Deformation of track components







# Spike Killing



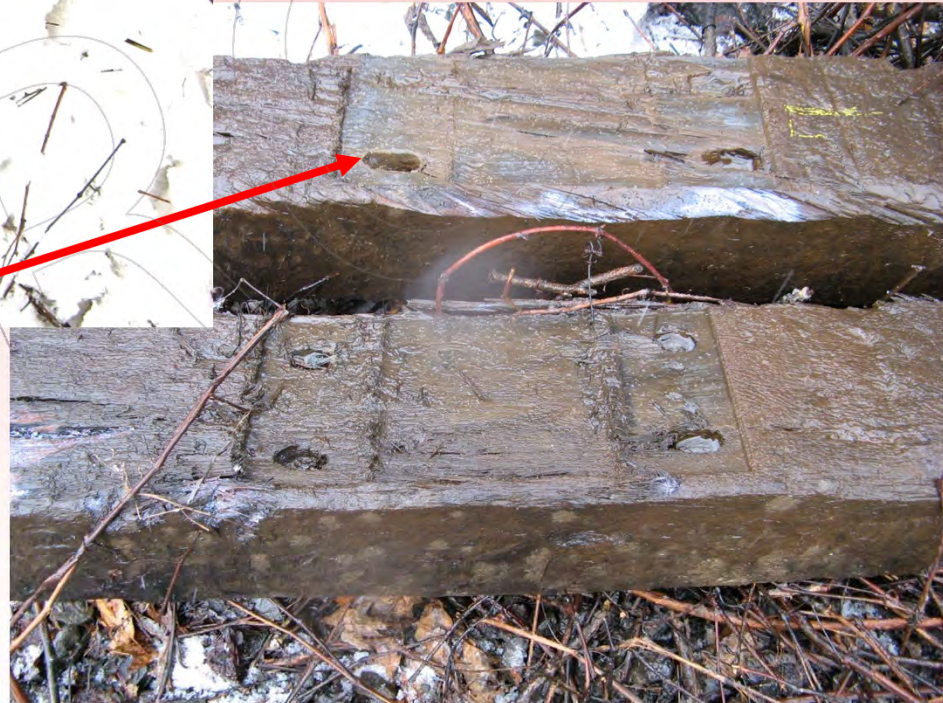
**Elongated Field side  
Spike hole and showing  
Loose spike in hole**



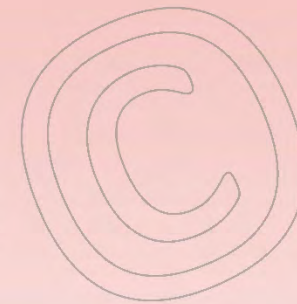
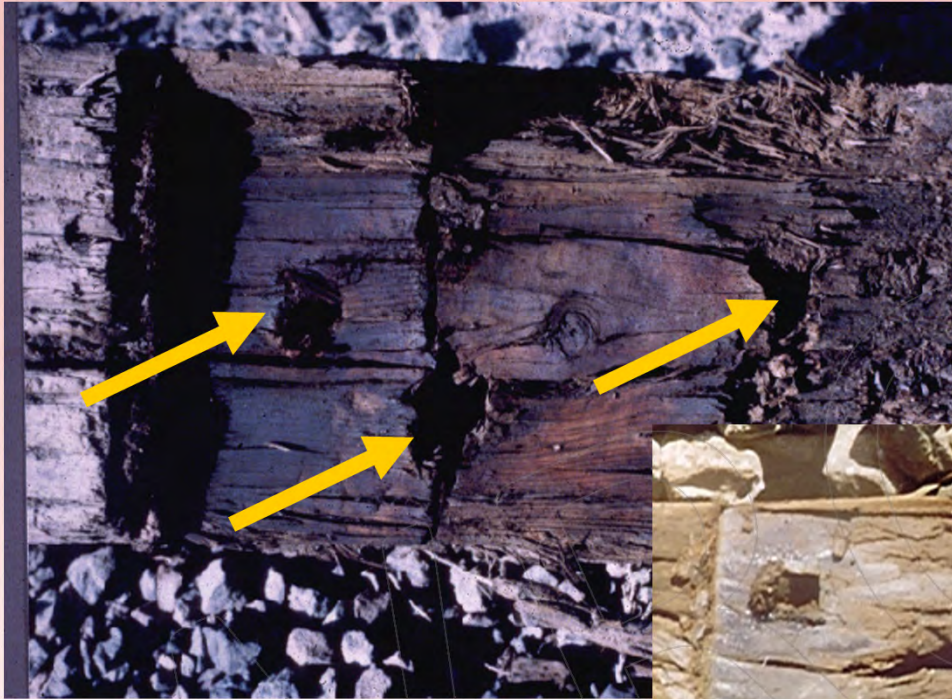




**Elongated  
Spike holes**





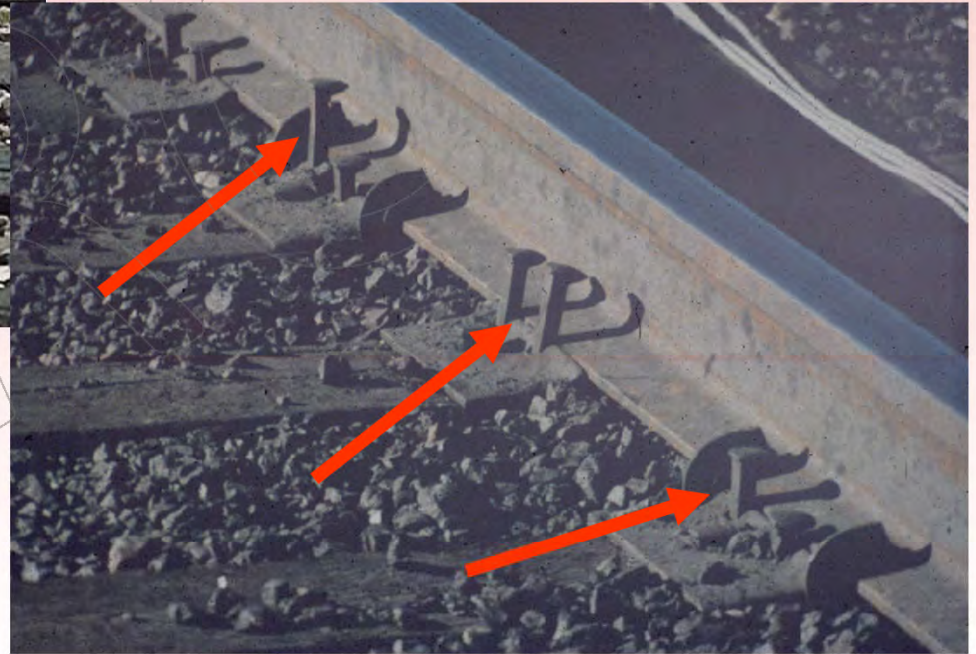


**Spike Kill**





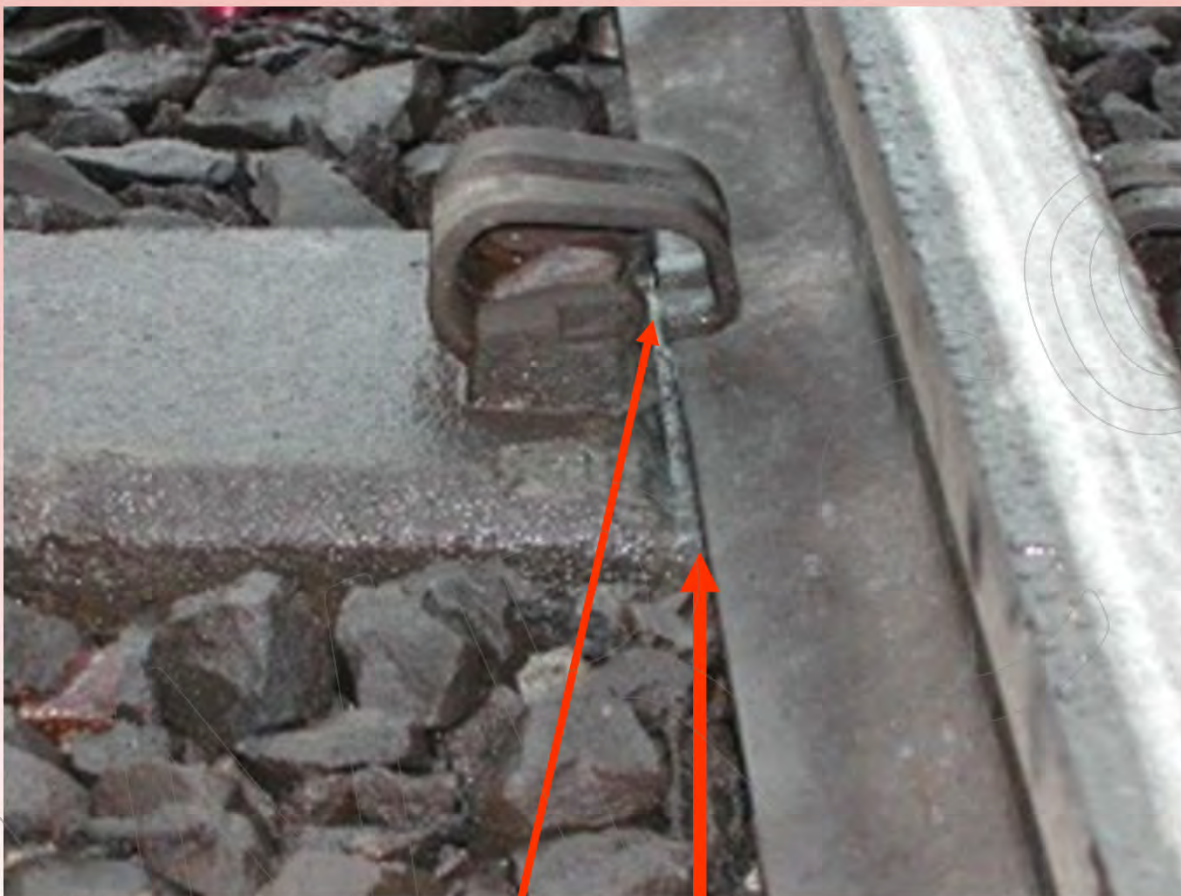
# Raised Spikes











**Watch for Tie Seat Abrasion under Rail**

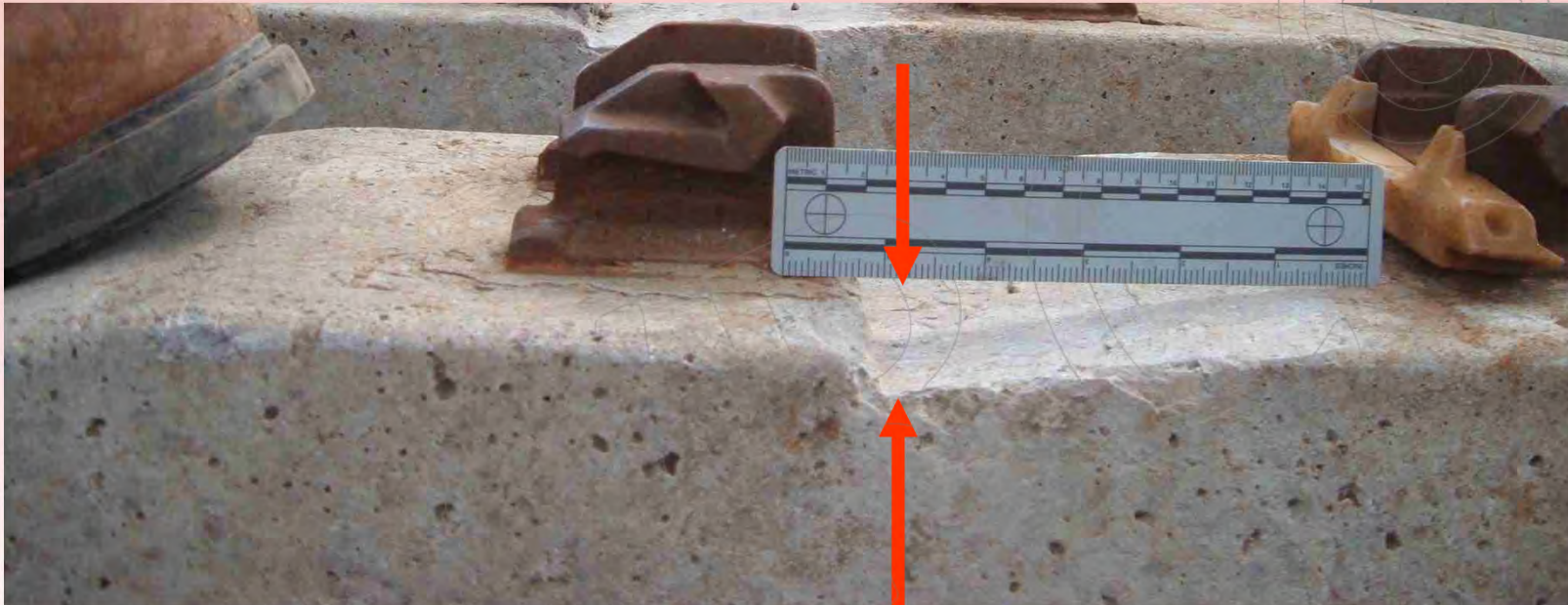




**Watch for Tie Seat Abrasion  
under Rail**



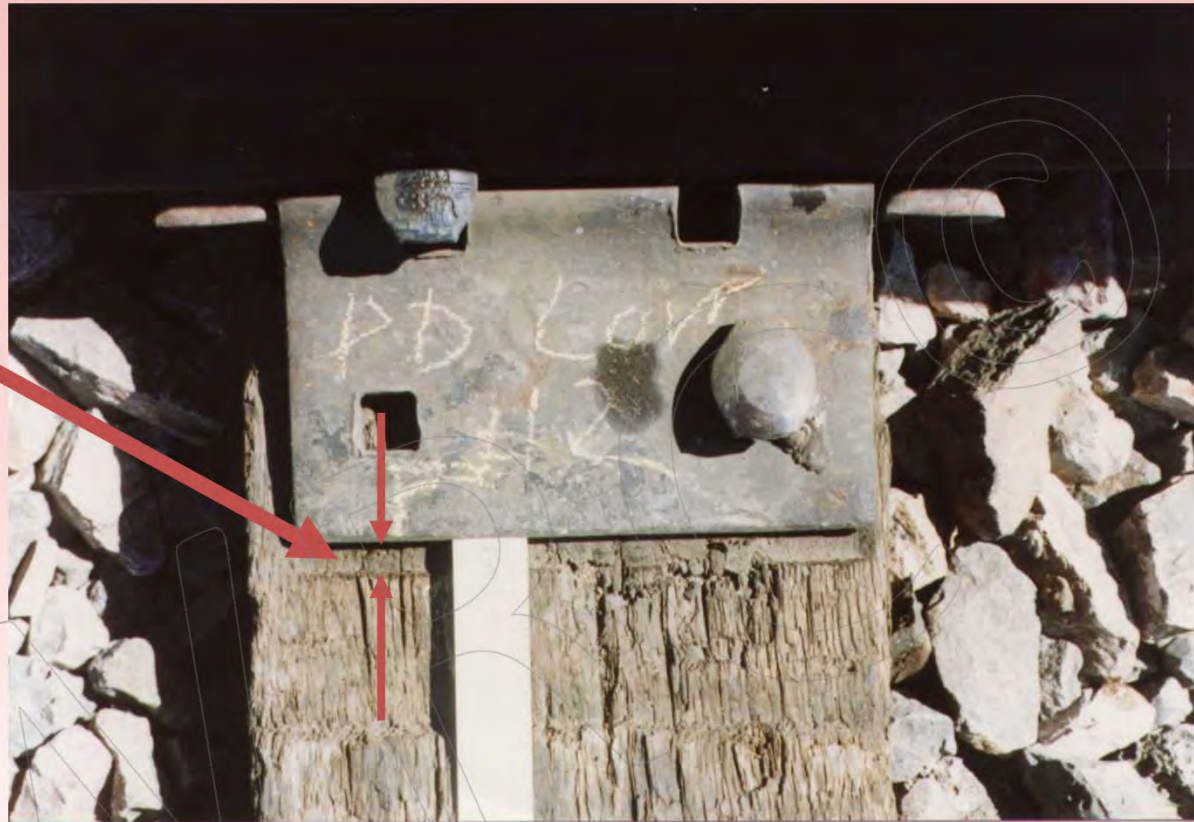




**Tie Seat Abrasion under Rail and Pad**



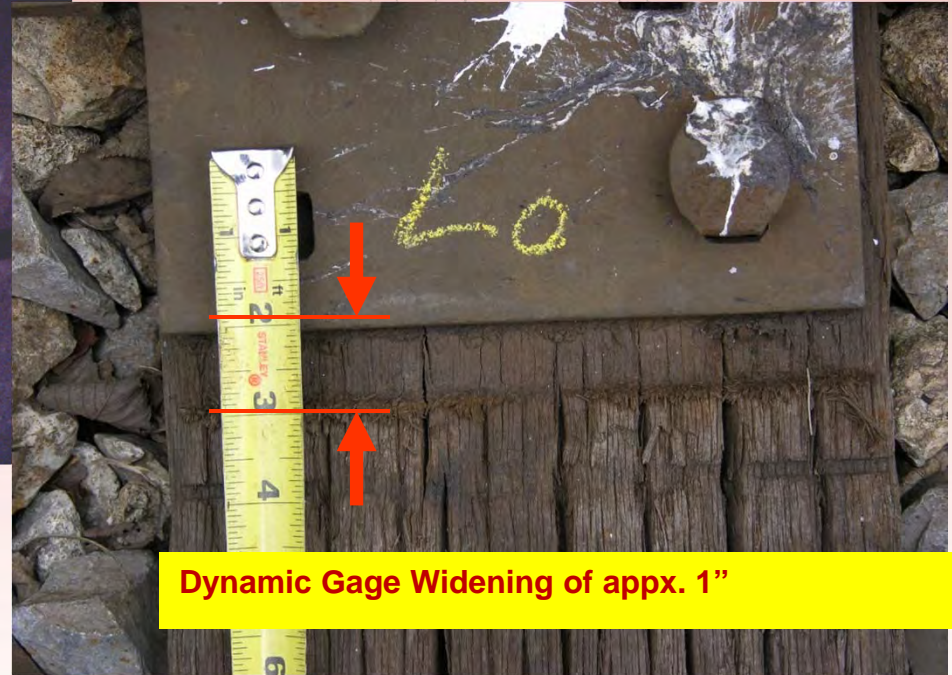
**Dynamic  
Gage  
Widening**



**No more than 1/2" lateral movement per FRA**







## Dynamic Gage Widening

No more than 1/2" lateral movement per FRA



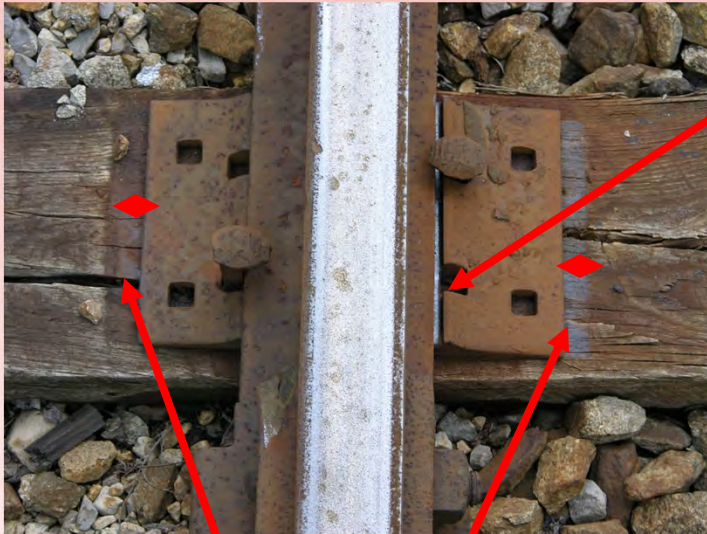
**Closeup showing amount of dynamic plate movement**





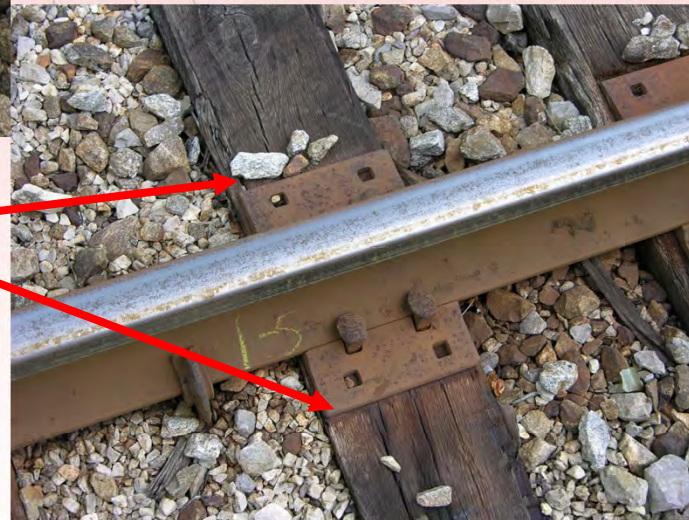






Note wear in shoulders of tie plate allowing additional rail movement under dynamic load.

Note significant dynamic gage widening under load.





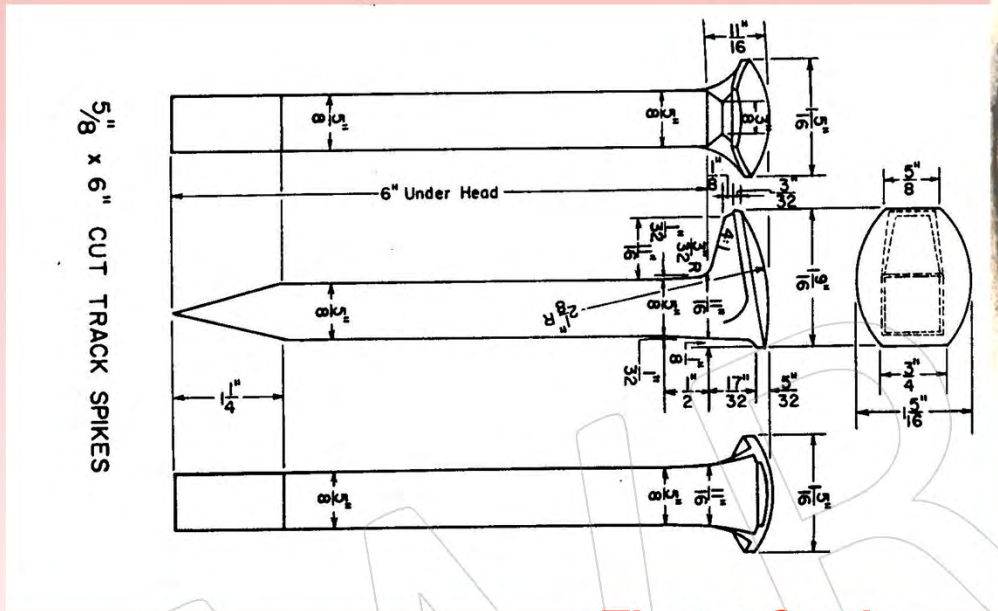
# Broken Fasteners

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# Cut Spike

## Sheared Spikes



Throat Cutting



Sheared Screw Spikes







Note wear from working against edge of tie plate

Random broken spikes from vicinity of POD. Note fatigue, rust, and polish on surface indicating not brand new break.





Fracture surfaces on three random broken spikes in Vicinity of POD. Note breaks appear to be “old”.







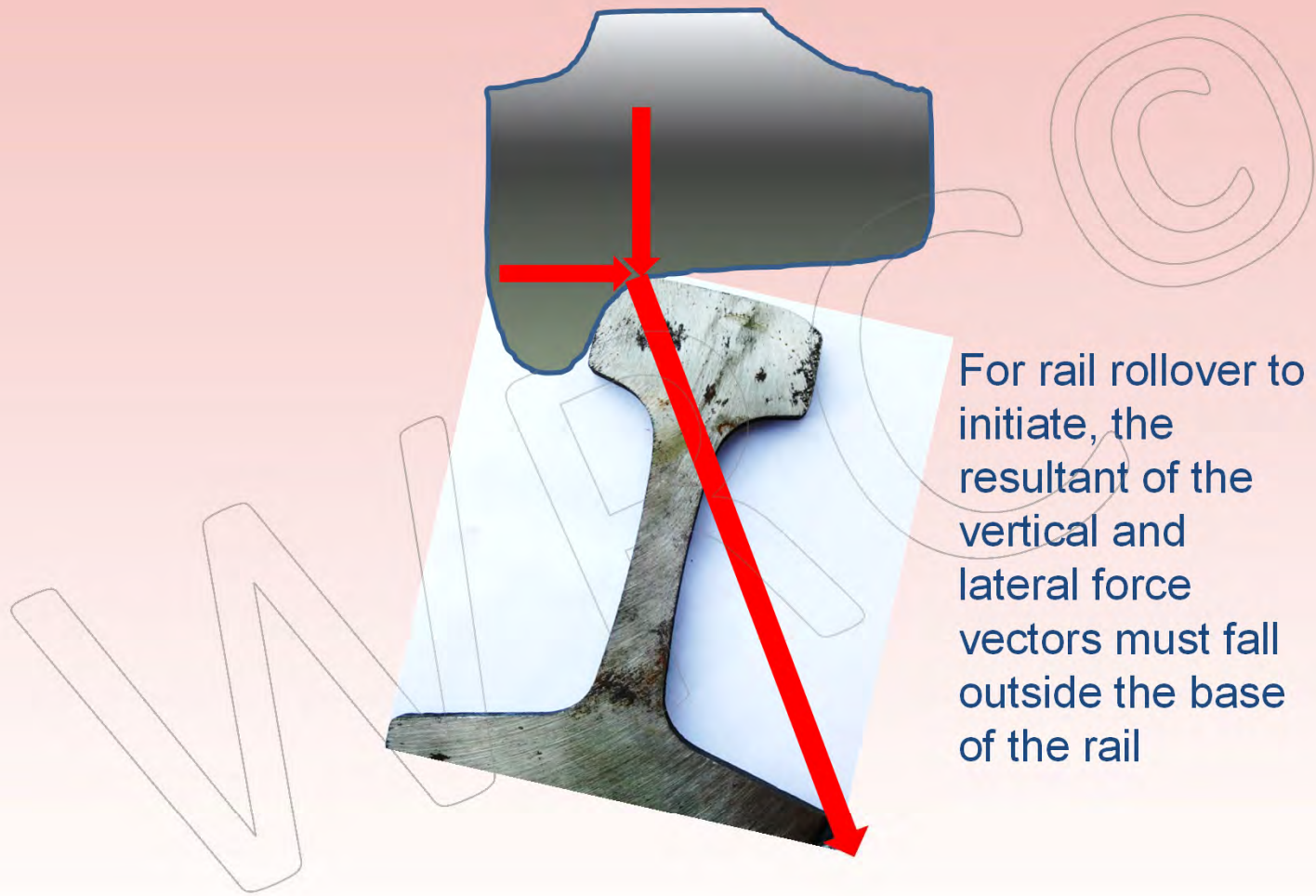
Random broken cut spikes and screw spikes from wide gage derailment. Note fatigue, rust, and polish on surface indicating not a brand new break.



# Why rail rolls over

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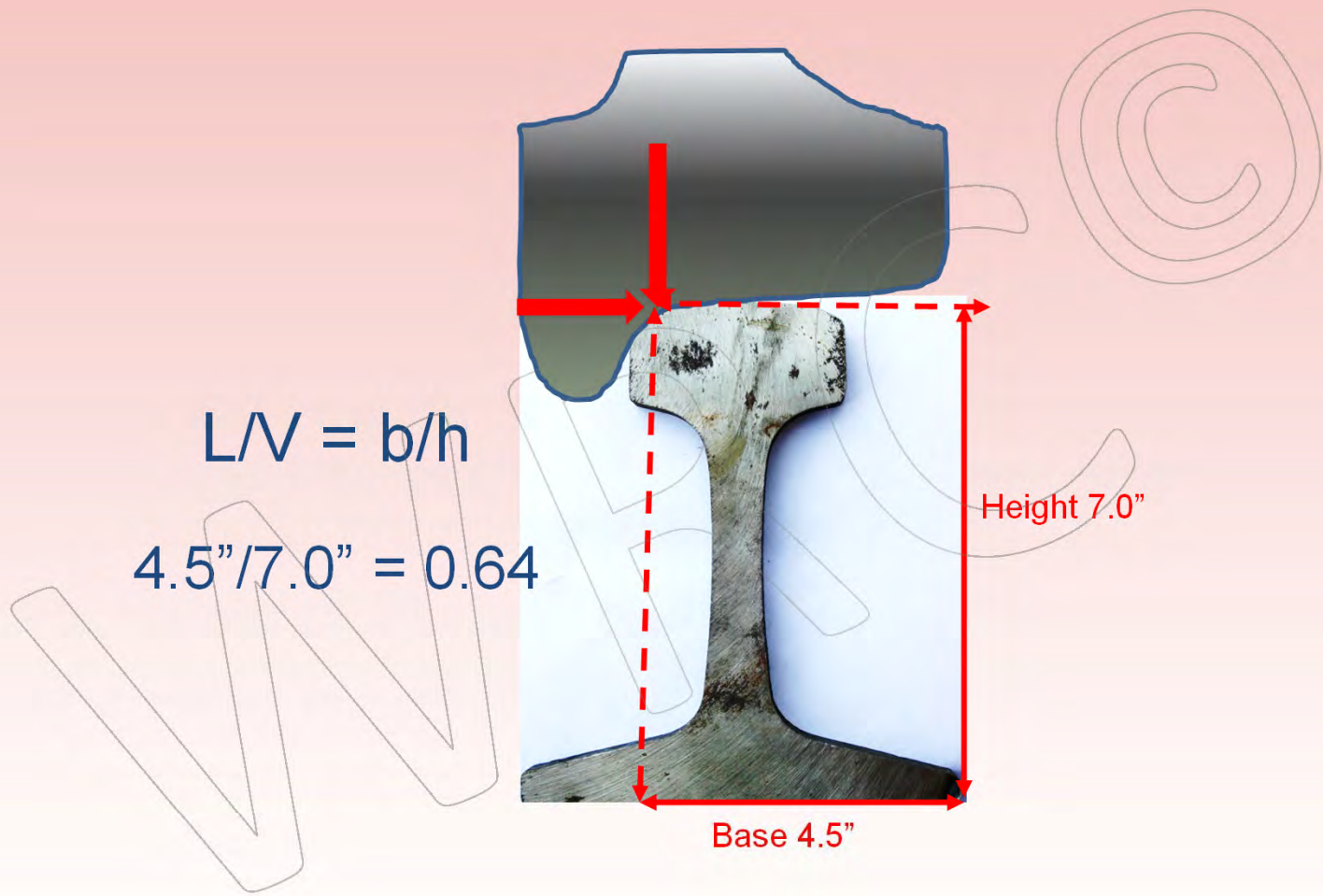




For rail rollover to initiate, the resultant of the vertical and lateral force vectors must fall outside the base of the rail

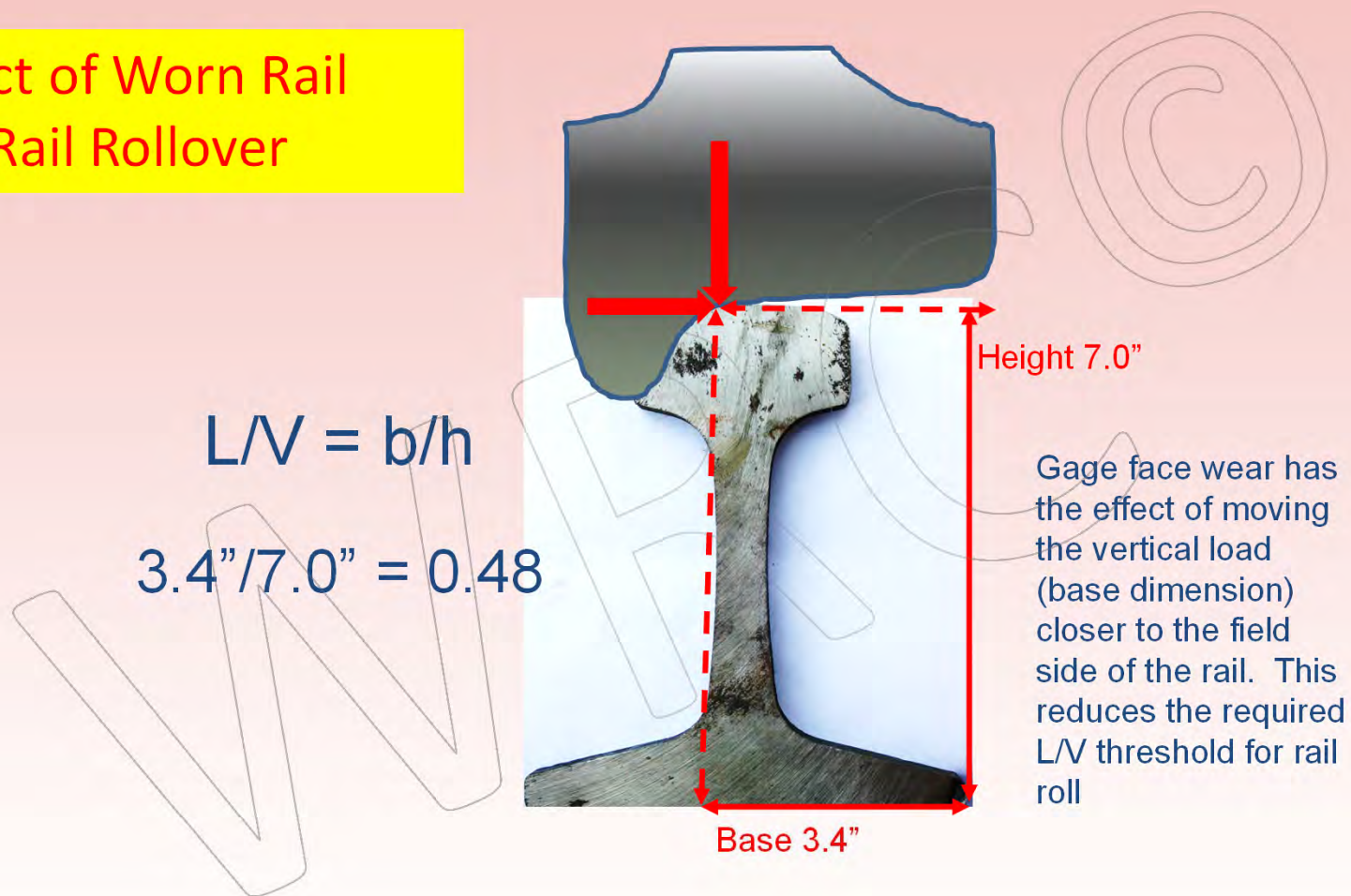


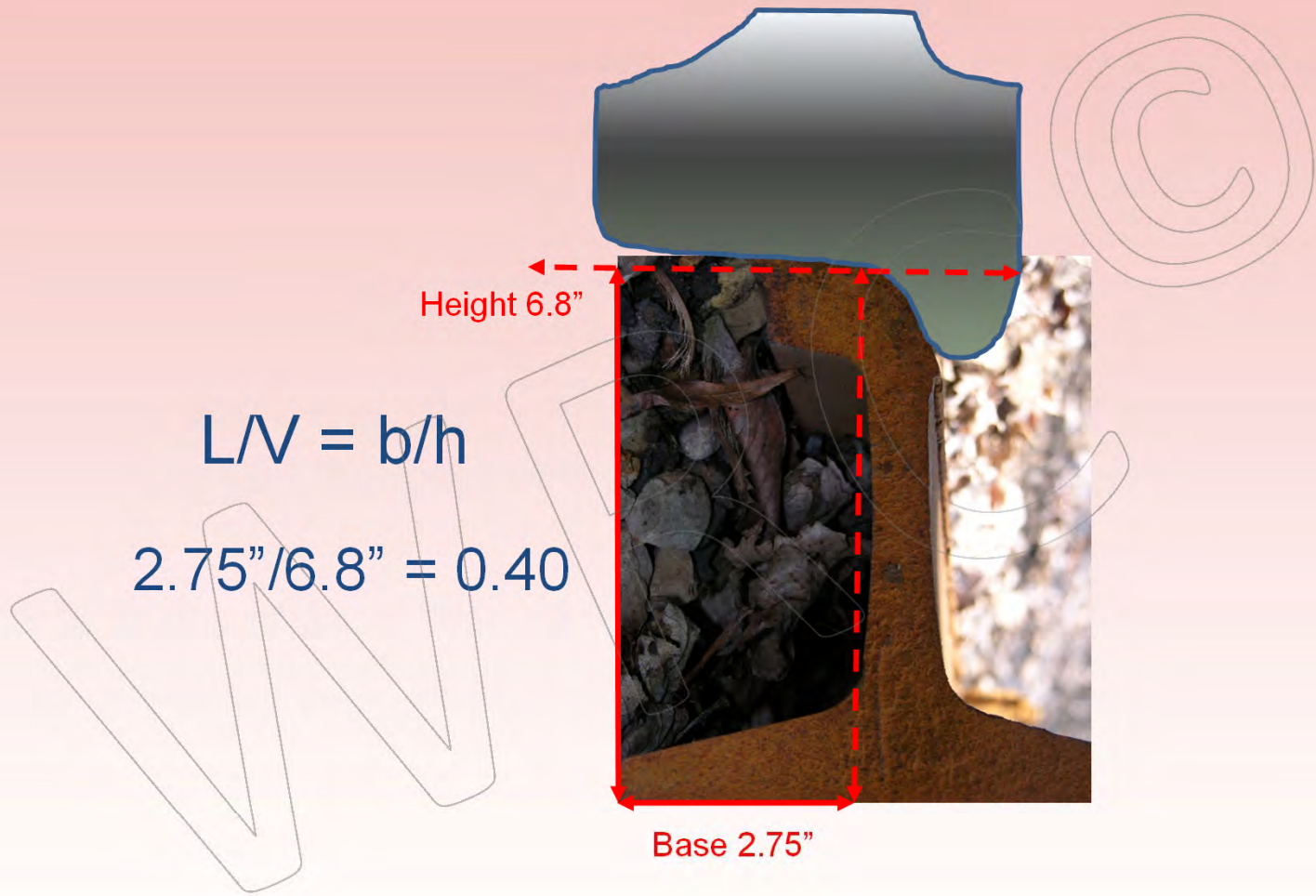




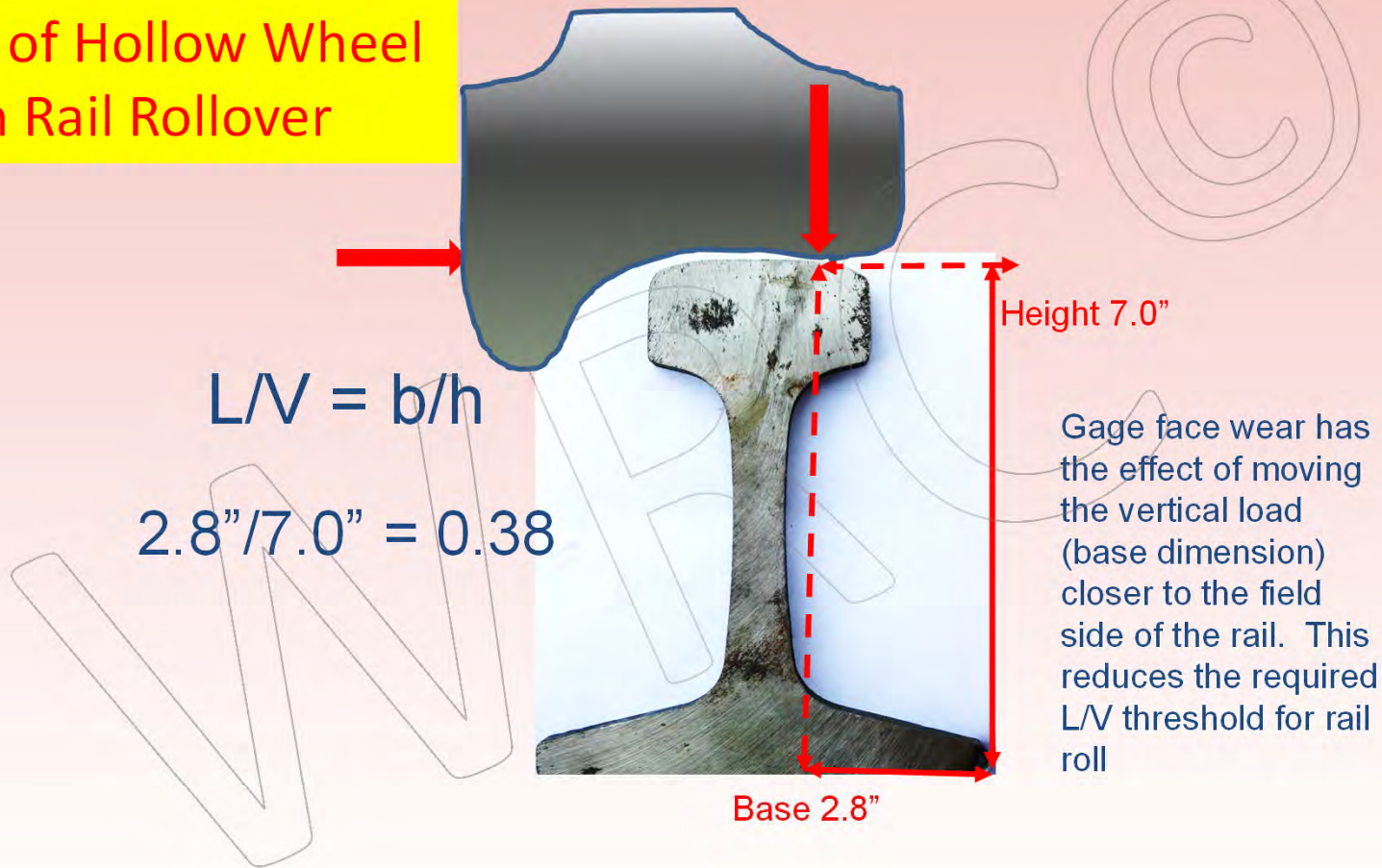


## Effect of Worn Rail on Rail Rollover



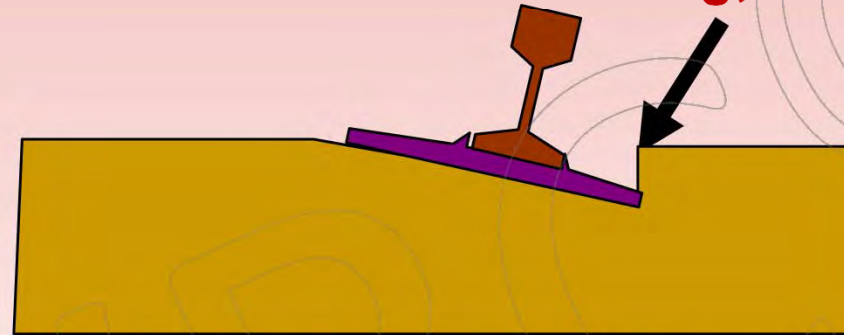


## Effect of Hollow Wheel on Rail Rollover





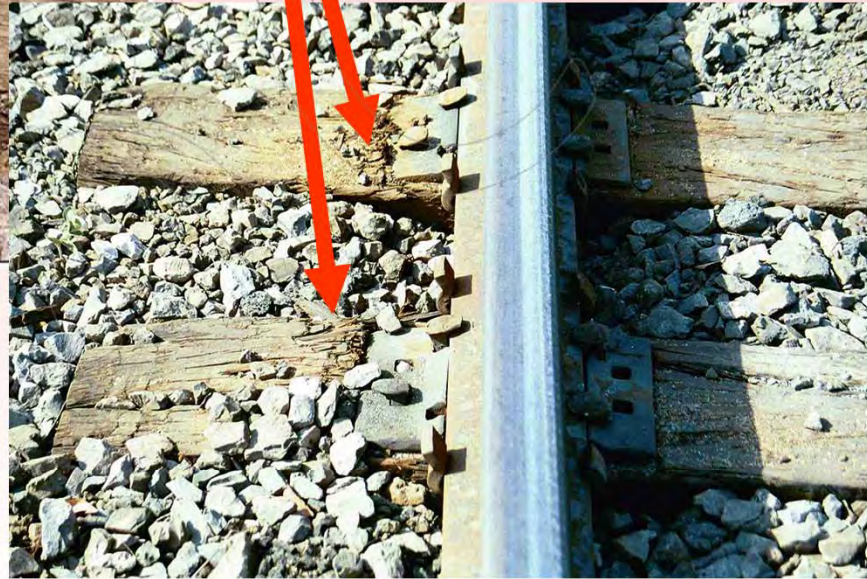
## Differential Plate Cutting; rail cant





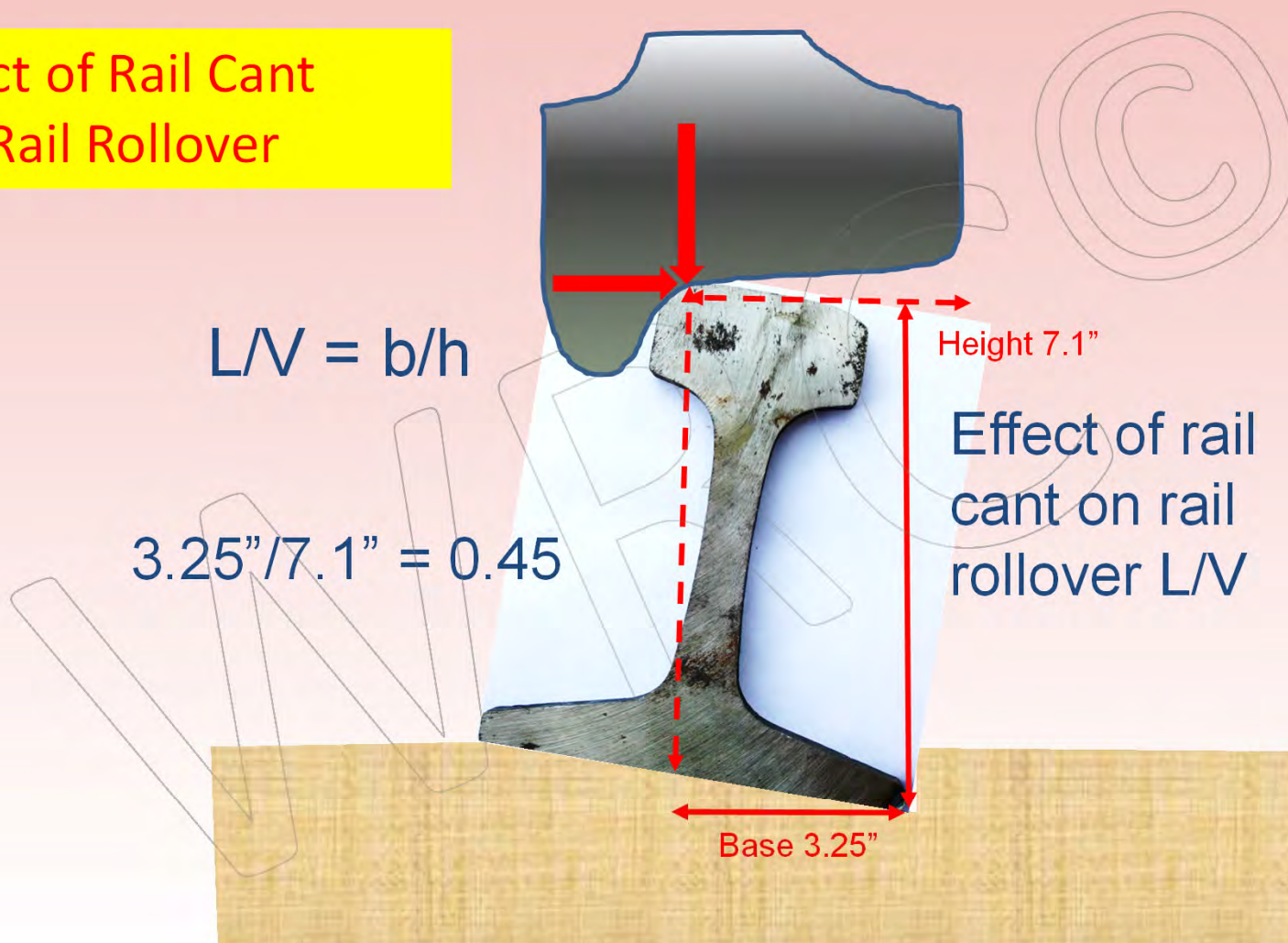


# Field Side Plate Cutting

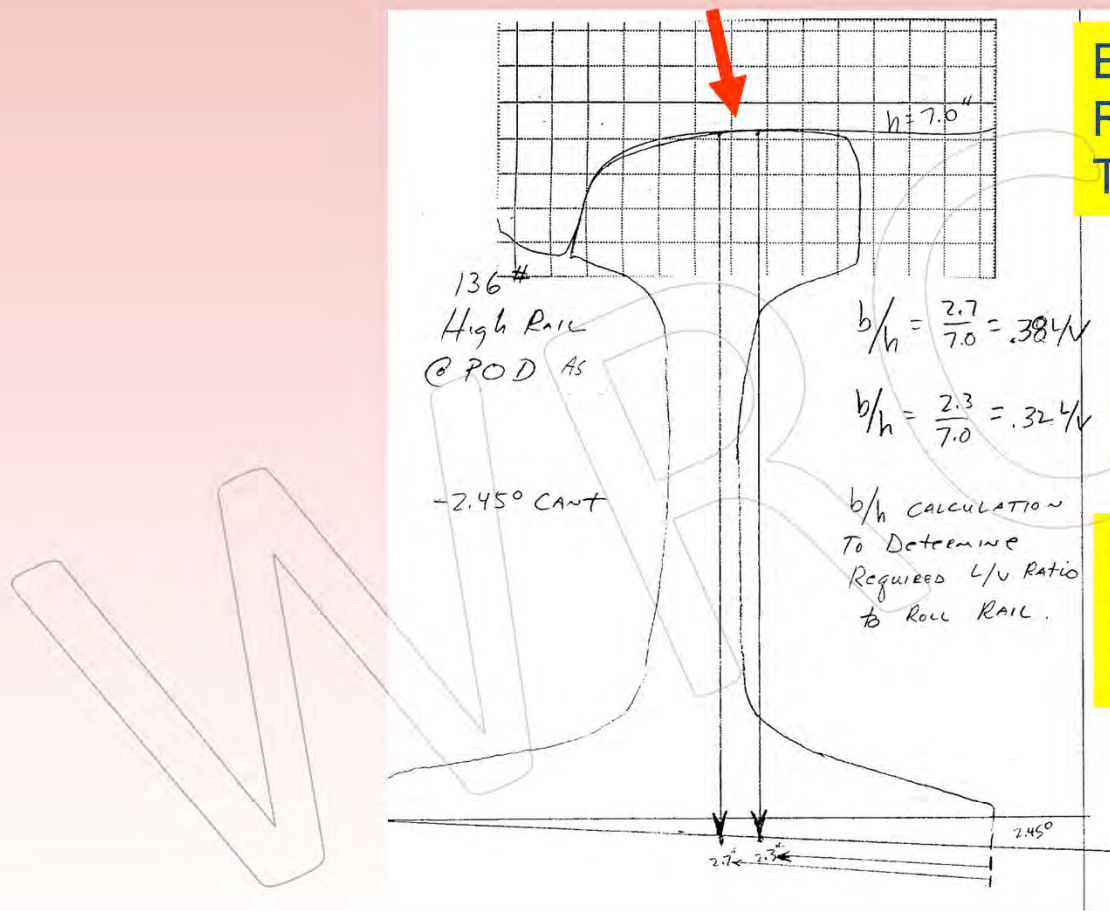




## Effect of Rail Cant on Rail Rollover

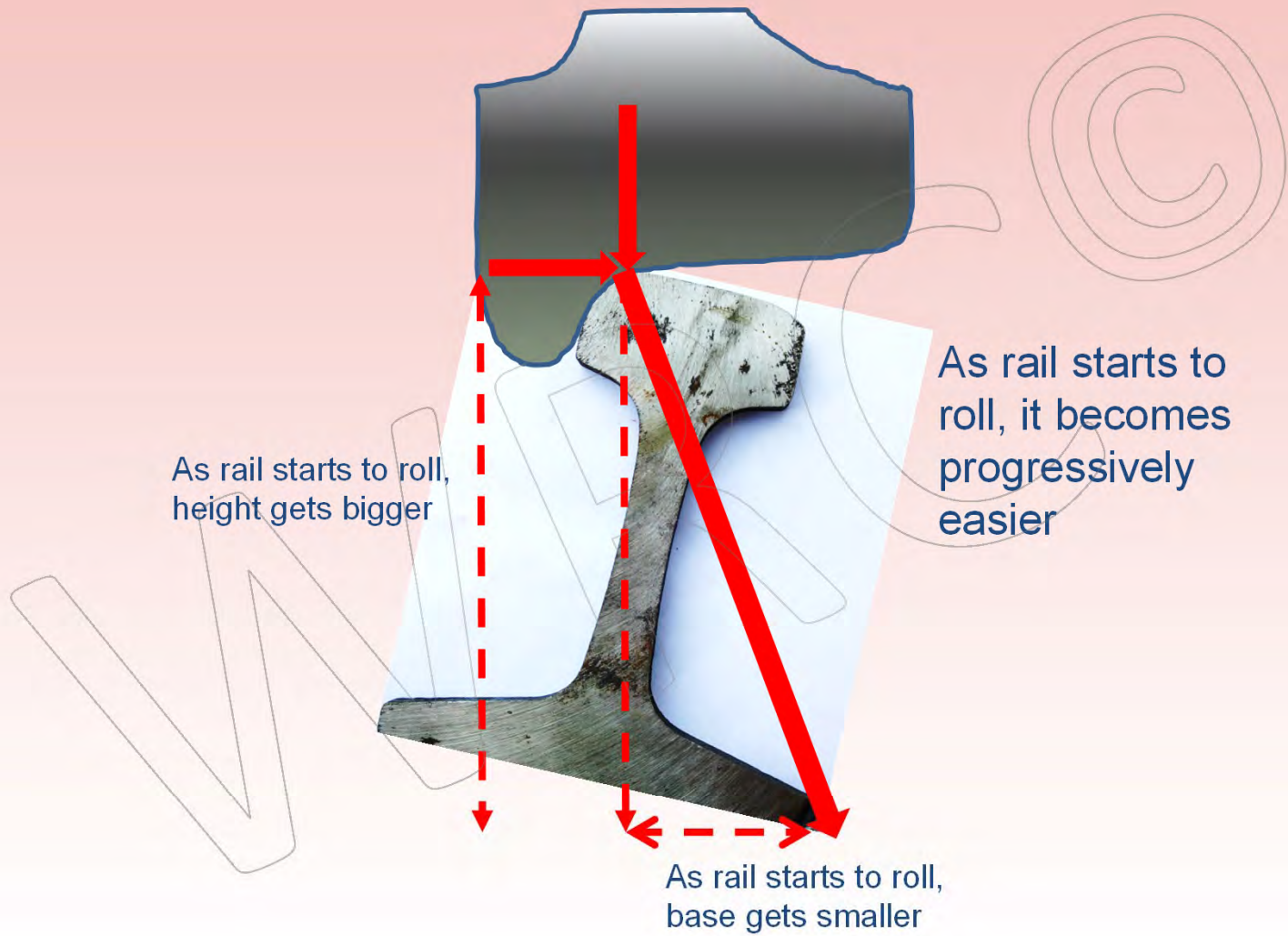


Effect of Negative Rail Cant and Two-point Contact

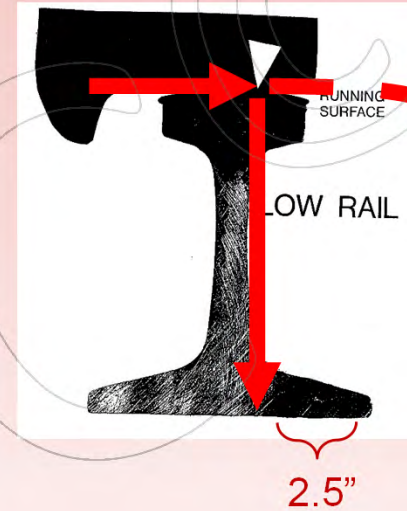


Lowers L/V ratio  
For rollover to  
0.38 – 0.32









**Why Low Rail is More Prone to Rail Rollover Especially with Hollow worn wheels**

B/h Ratio

2.5"/7.0"

L/V for rollover = 0.35



How maintenance practices, particularly incorrect restoration of rail cant, and lack of grinding can affect the steering forces developed





Poor Steering; Reverse cant on both rails. No grinding







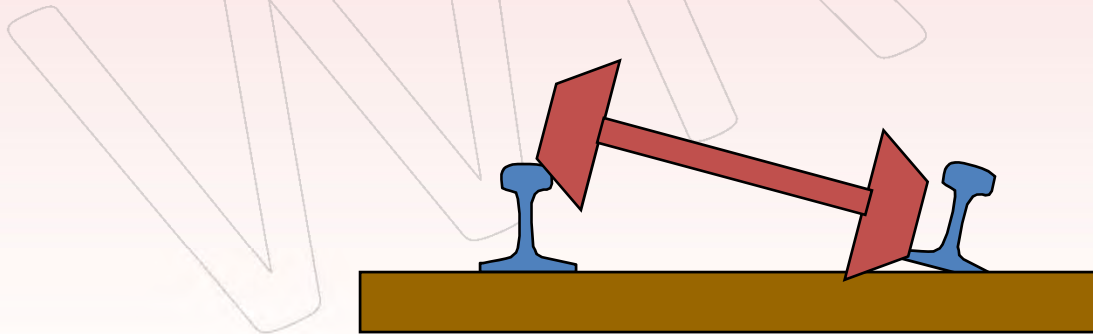
Very Poor Steering; High Rail  
Positive Cant restored; Low rail  
still has reverse cant. No  
grinding



# Wide Gage Derailments

## T110, T111 - Symptoms

- Wheels found dropping in the gage
- May find wheel Drop-in Mark on gage corner of rail at POD
- Pre-existing Wide Gage or Evidence of Dynamic Gage Widening
- Either Rail May be Rolled Outward

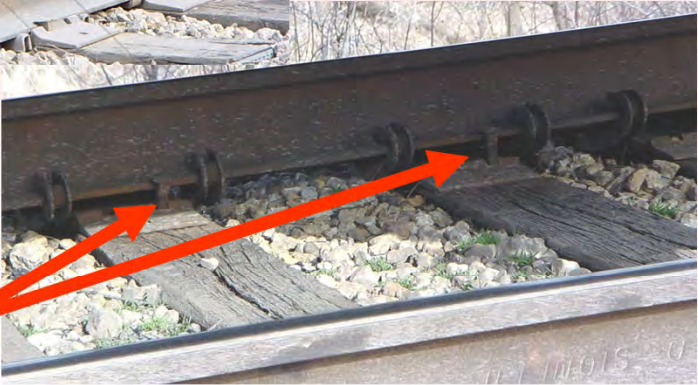


# Anatomy of a Wide Gage Derailment T110, T111

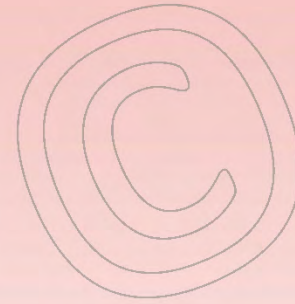
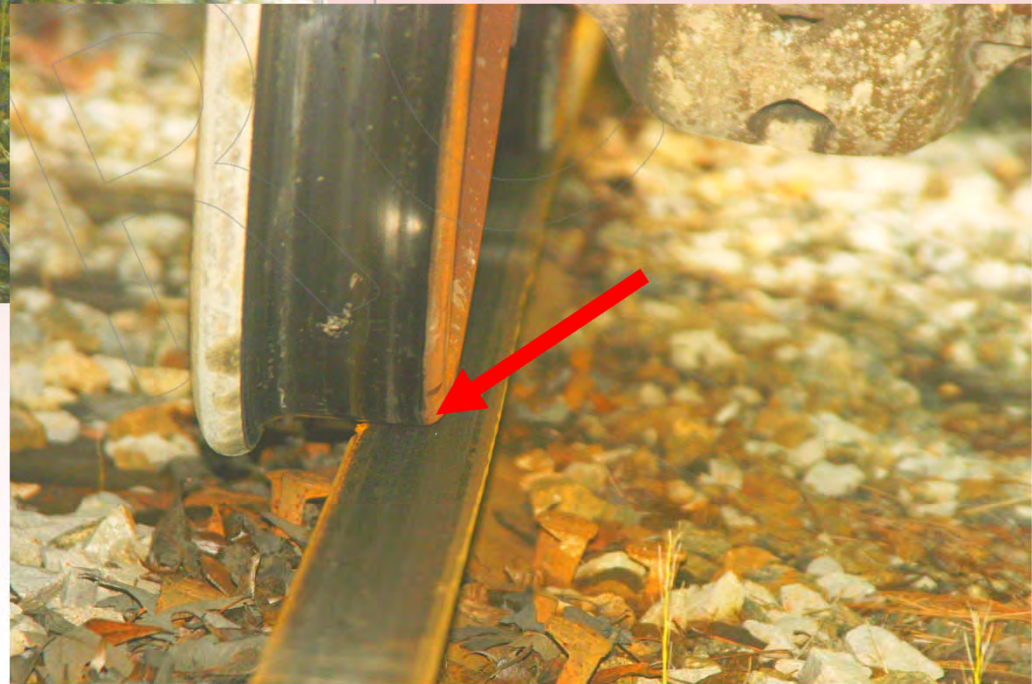
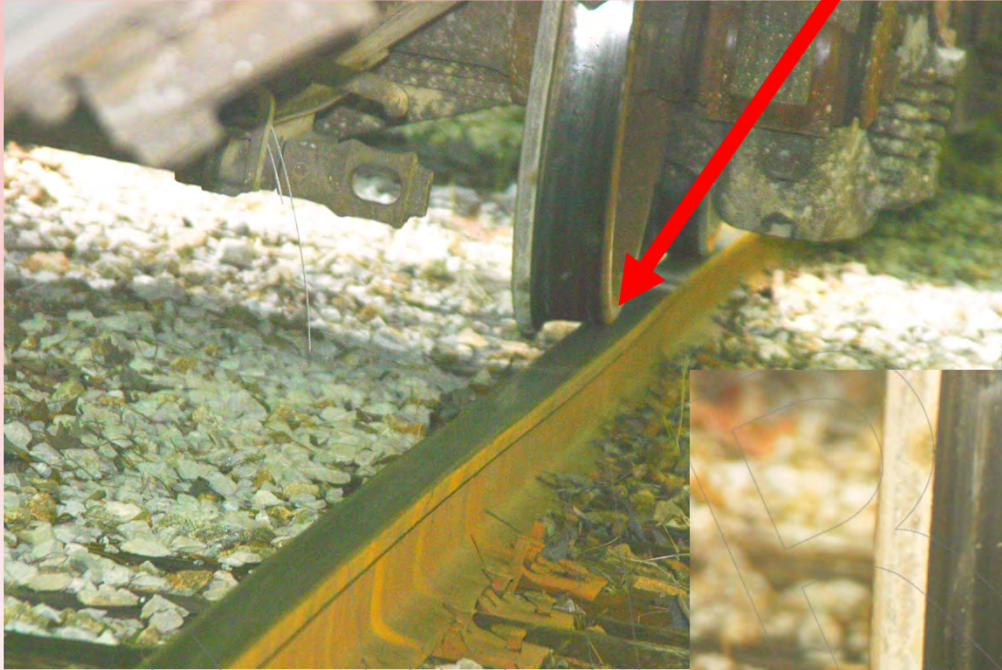


One or both rails  
rolled  
outward under train

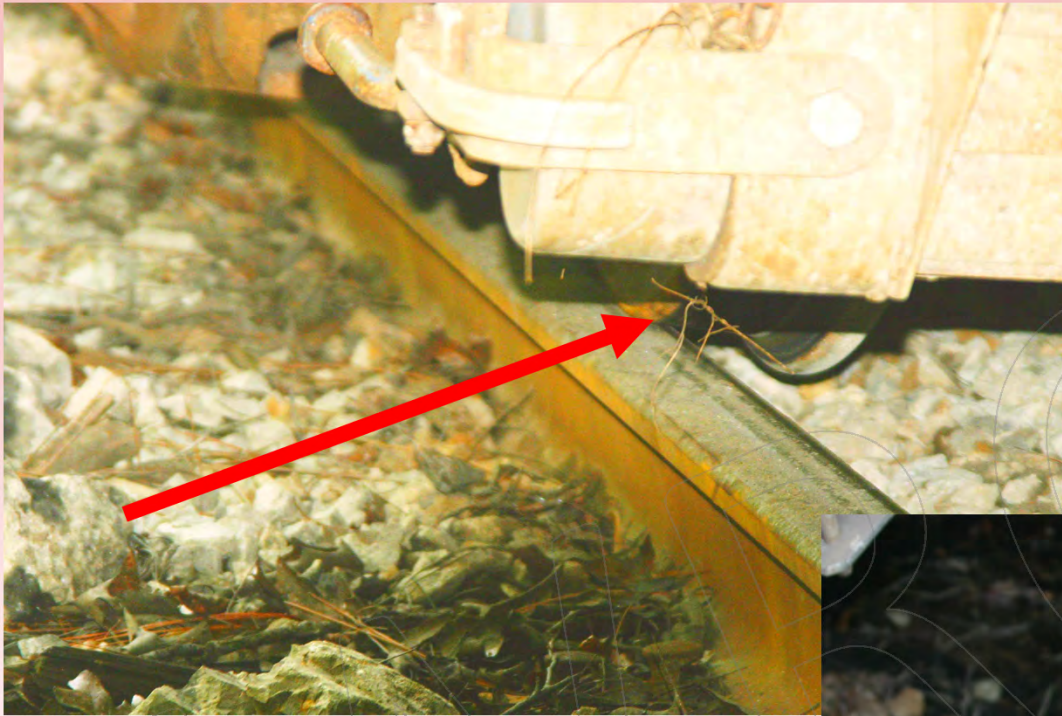
Evidence of raised  
spikes leading into  
POD



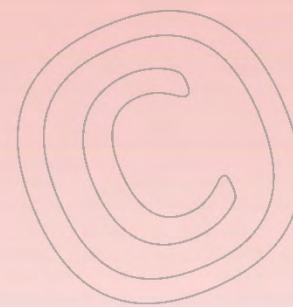
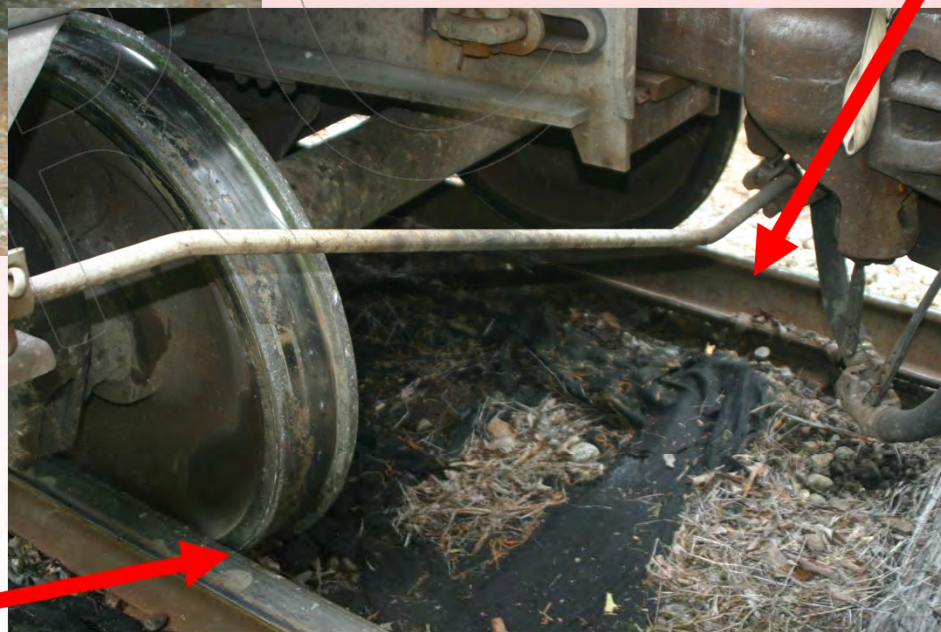
















# Wide Gage Summary

- Wide gage derailments involve complex interaction between the track, rail, vehicles, and operating parameters.
- The investigator must determine if the gage widened progressively or catastrophically.
- Each component of the system must be evaluated
  - Ties, fasteners, wheel-rail contact, wheelset steering, rail wear, track geometry, speed, wheel wear, vehicle condition, operating factors
- Only after a detailed investigation can a true root cause(s) be developed.
- Not all wide gage derailments are due to weak track!





# The End

